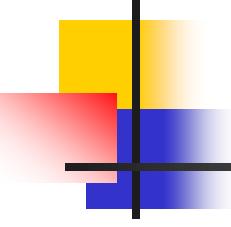


Multiaxial Fatigue

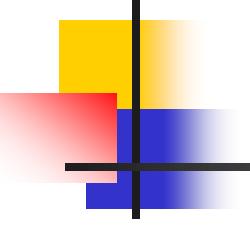
Professor Darrell Socie

© 2008-2014 Darrell Socie, All Rights Reserved



Outline

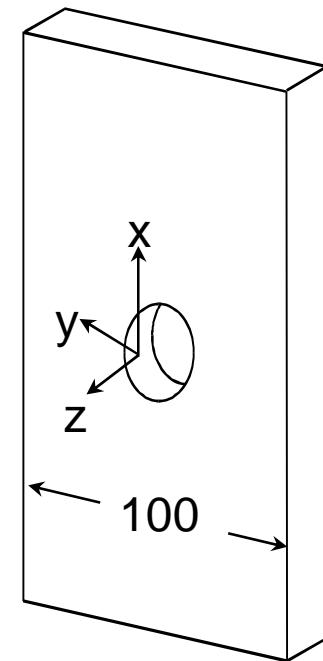
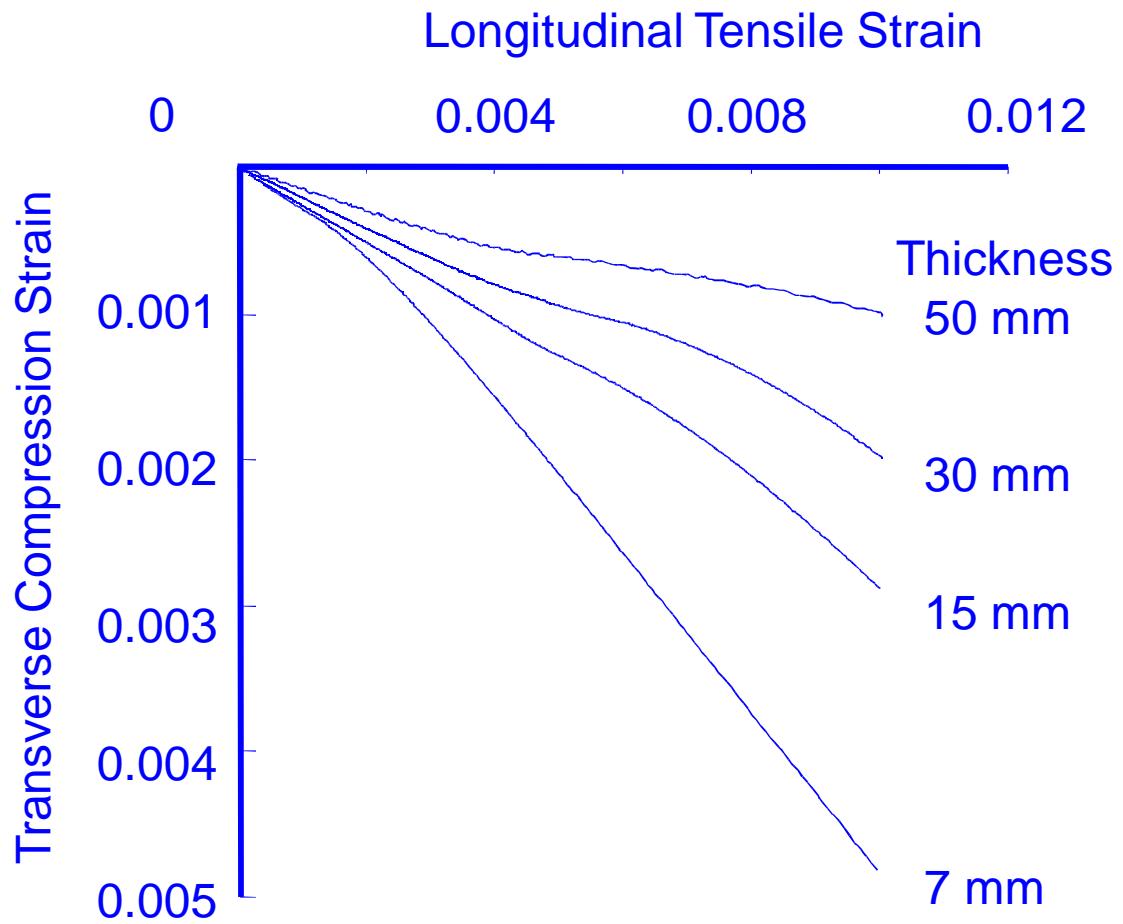
- Stresses around holes
- Crack Nucleation
- Crack Growth

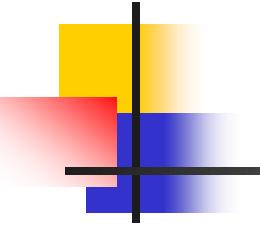


Multiaxial Fatigue Problems

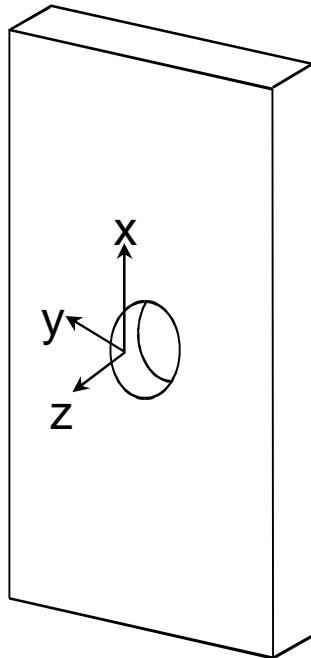
- Uniaxial loading that produces multi stresses and stress concentrators
- Multiaxial loading that produces uniaxial stresses around stress concentrators
- Multiaxial loading that produces multiaxial stresses around stress concentrators
- Multiaxial loading that causes mixed mode long crack growth

3D stresses

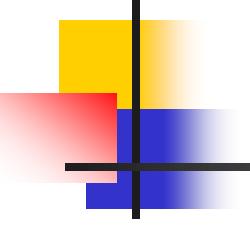




Notch Stresses



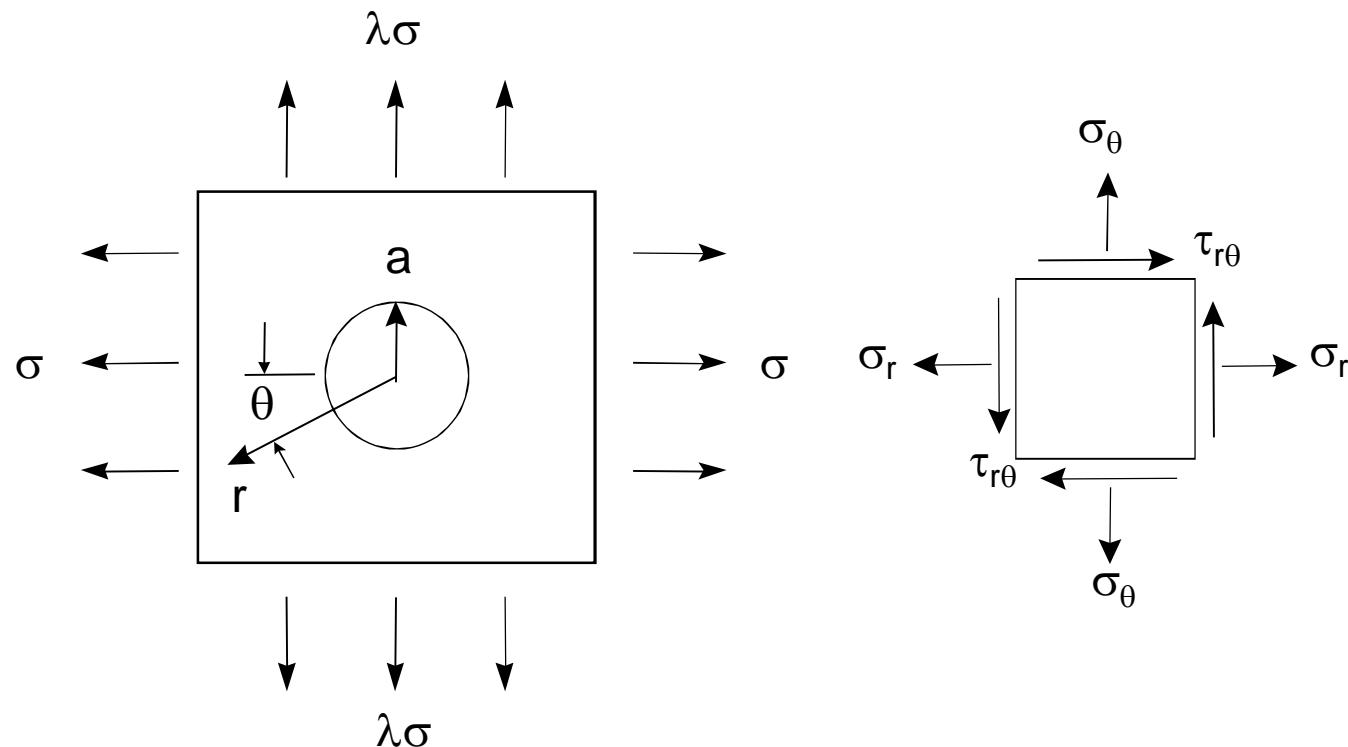
t	ε_x	ε_z	σ_x	σ_z
7	0.01	-0.005	63.5	0
15	0.01	-0.003	70.6	14.1
30	0.01	-0.002	73.0	21.8
50	0.01	-0.001	75.1	29.3



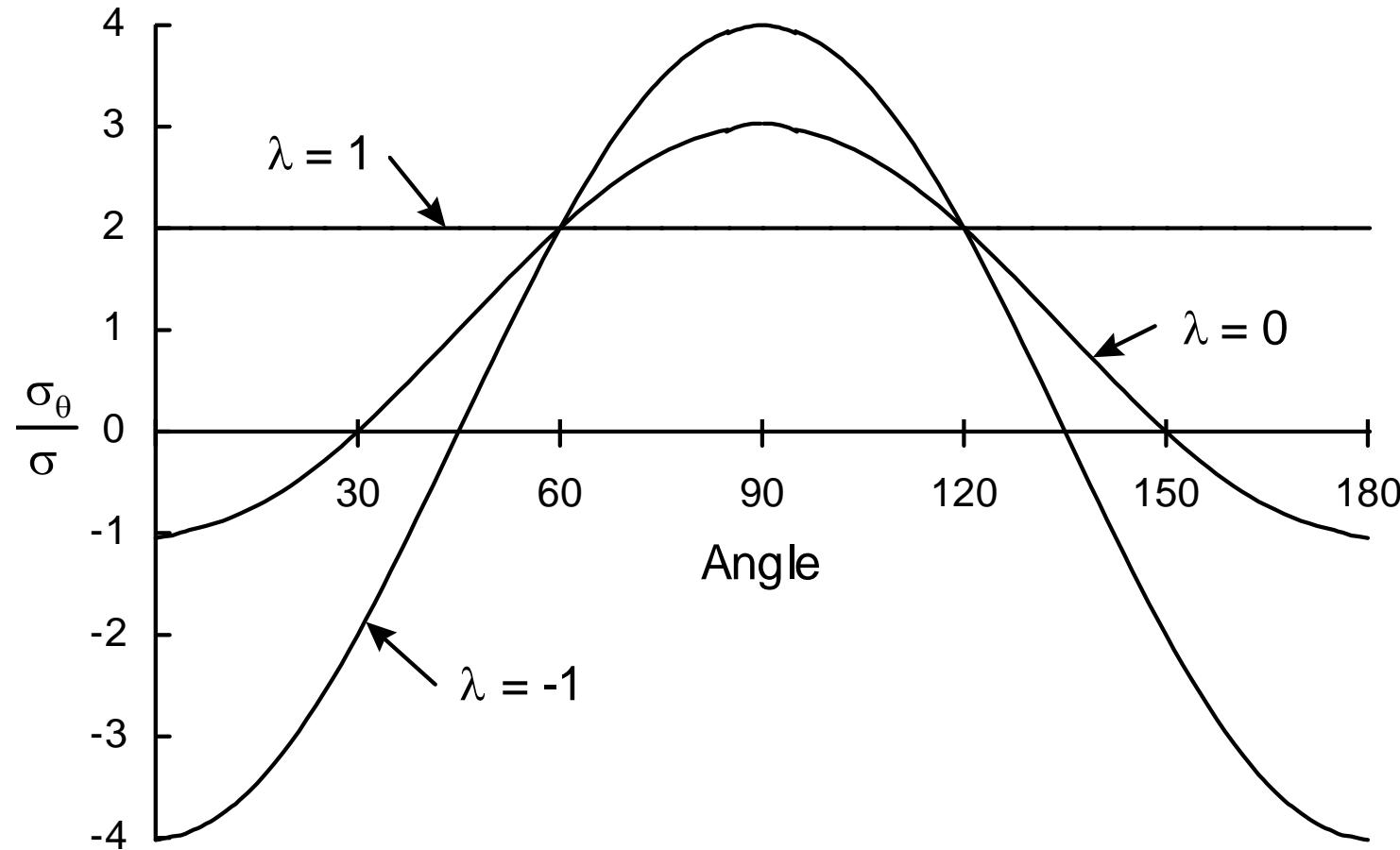
Multiaxial Fatigue Problems

- Uniaxial loading that produces multi stresses and stress concentrators
- Multiaxial loading that produces uniaxial stresses around stress concentrators
- Multiaxial loading that produces multiaxial stresses around stress concentrators
- Multiaxial loading that causes mixed mode long crack growth

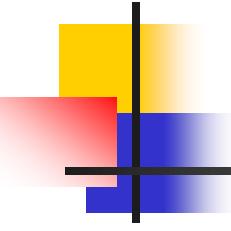
Hole in a Plate



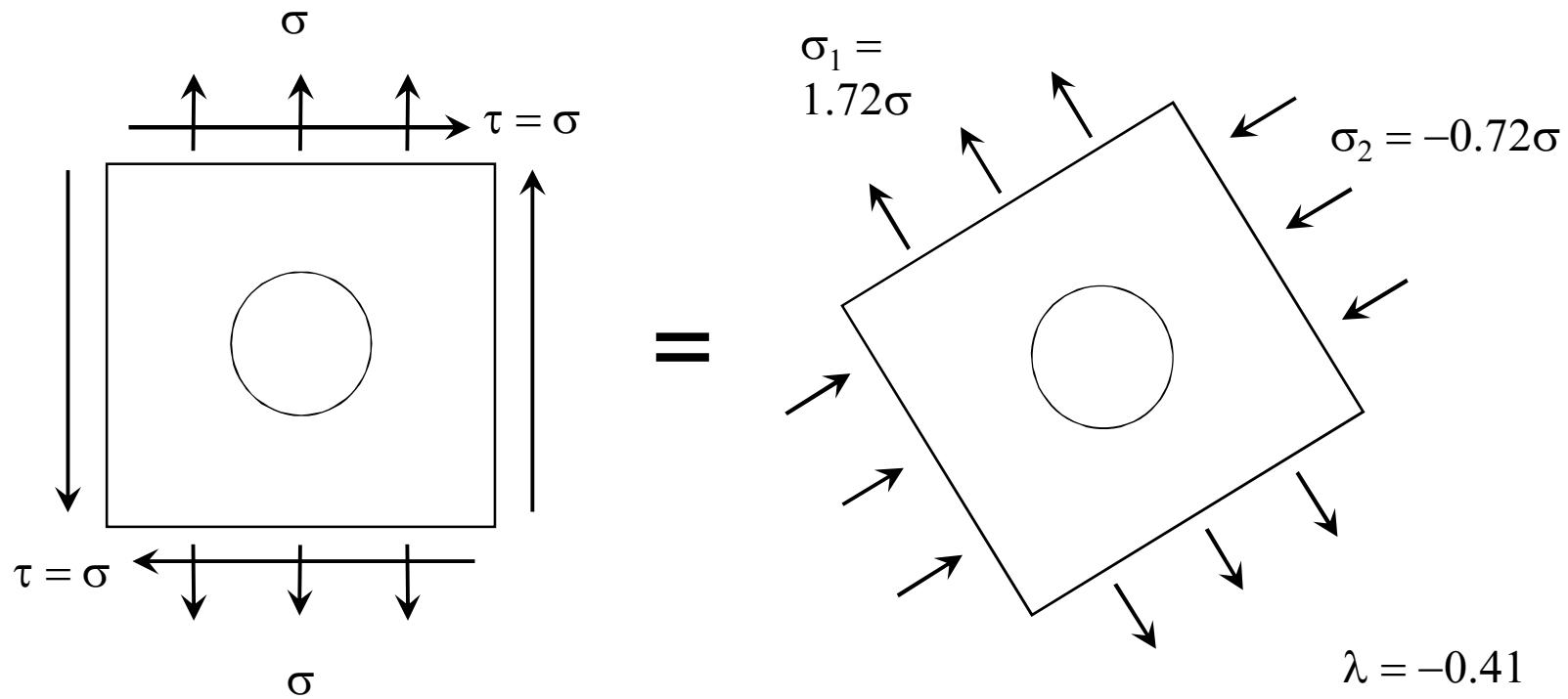
Stresses at the Hole



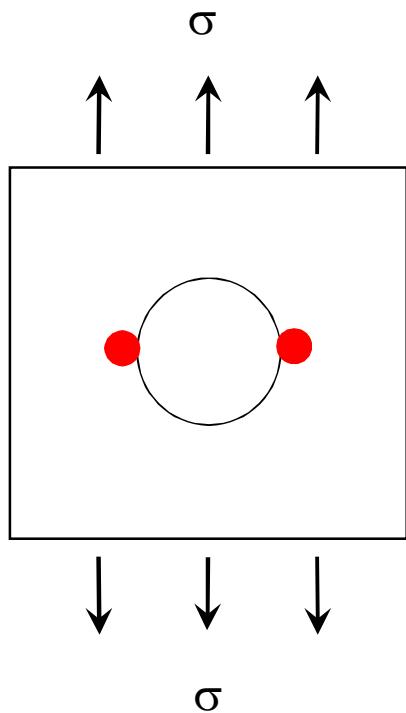
Stress concentration factor depends on type of loading



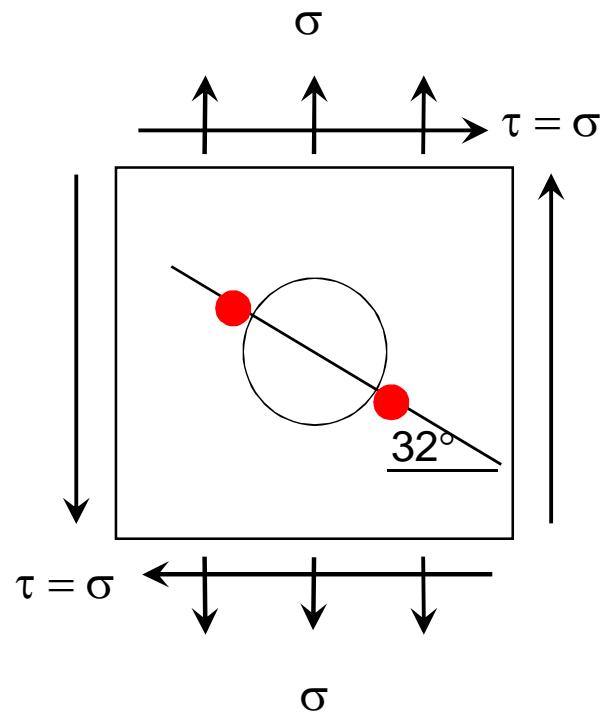
Combined Loading



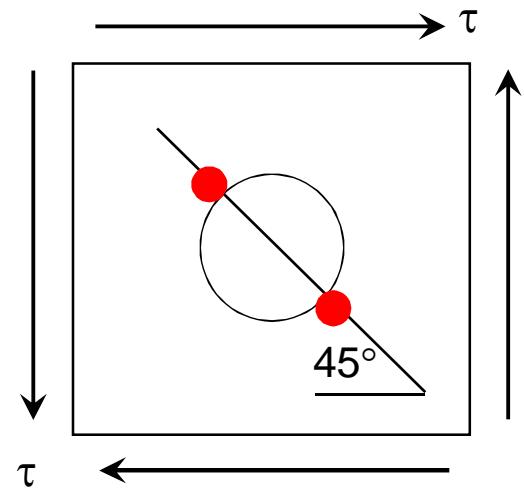
Maximum Tensile Stress Location



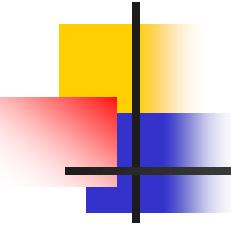
$$K_t = 3$$
$$\sigma_1 = \sigma$$



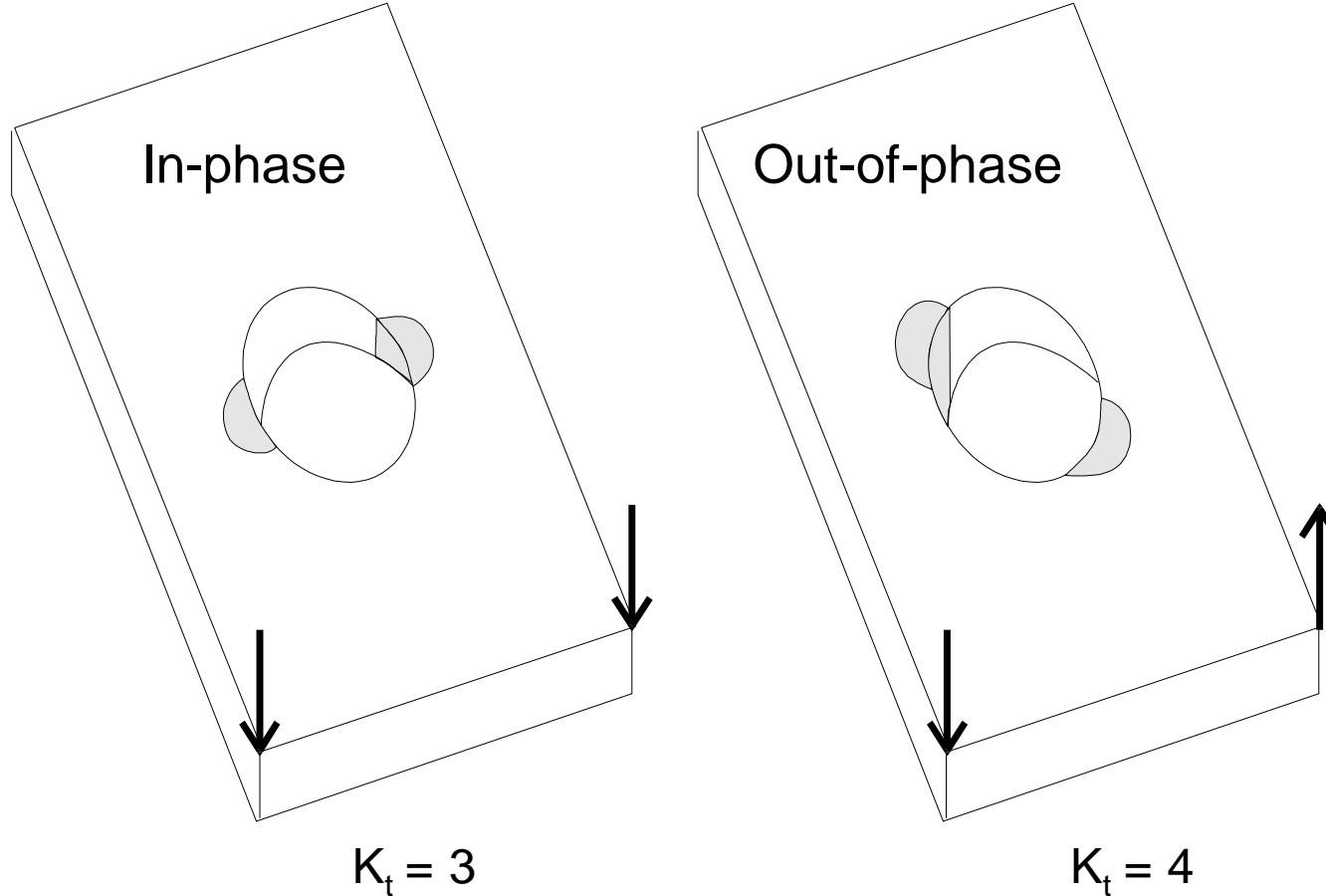
$$K_t = 3.41$$
$$\sigma_1 = 1.72\sigma$$



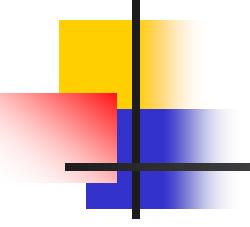
$$K_t = 4$$
$$\sigma_1 = \tau$$



In and Out of Phase Loading

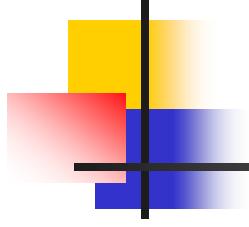


Damage location changes with load phasing

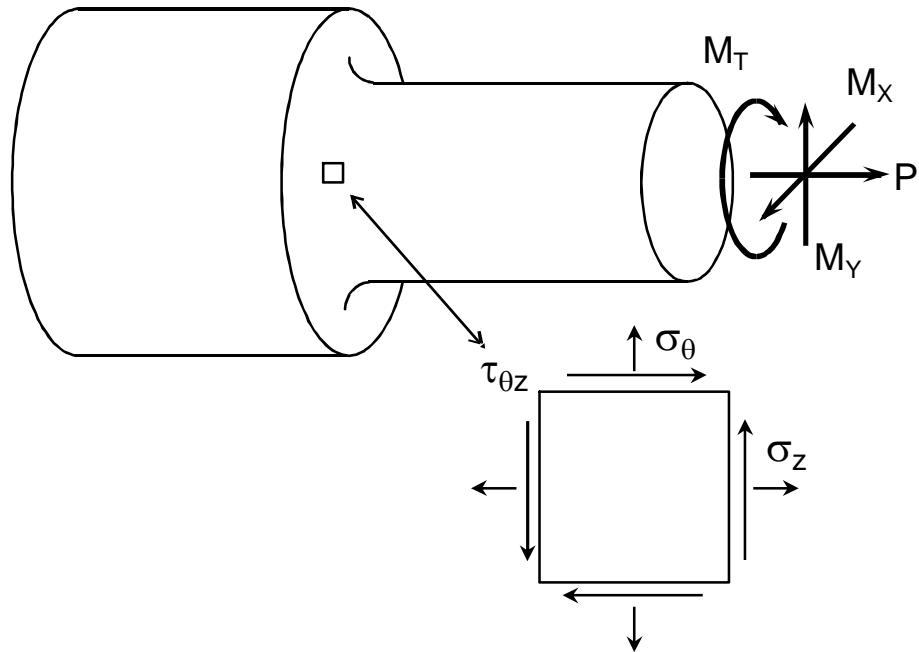


Multiaxial Fatigue Problems

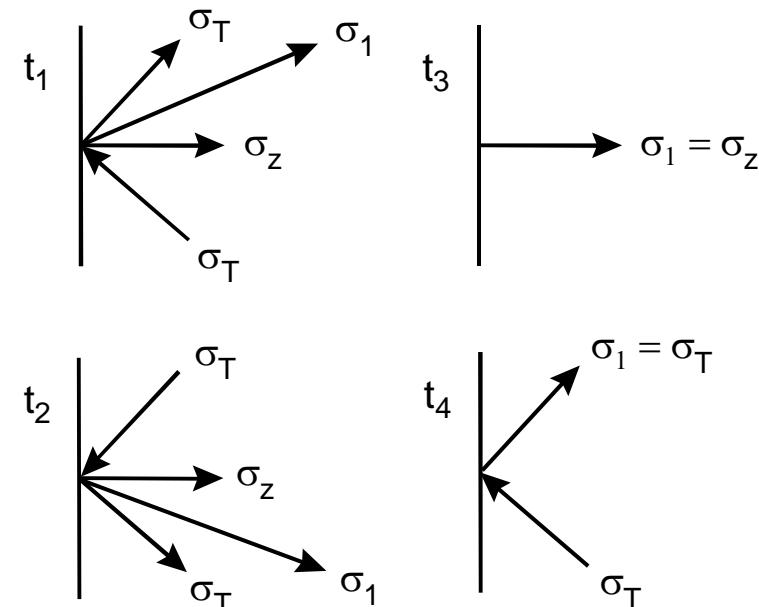
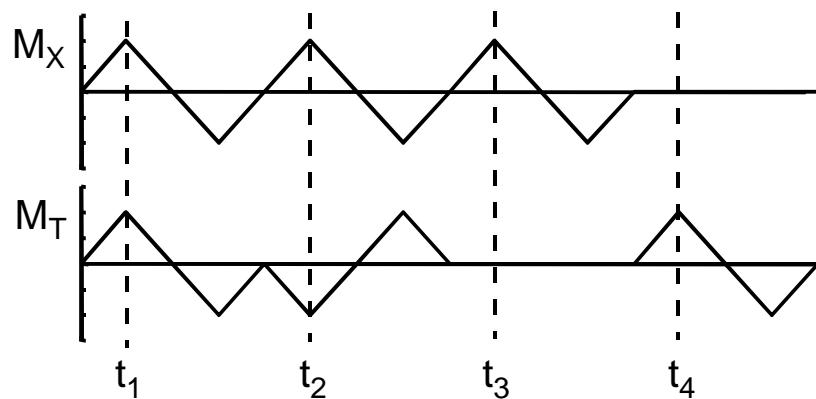
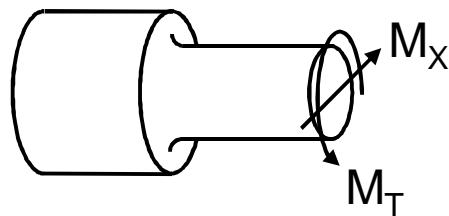
- Uniaxial loading that produces multiaxial stresses around stress concentrators
- Multiaxial loading that produces uniaxial stresses around stress concentrators
- **Multiaxial loading that produces multiaxial stresses around stress concentrators**
- Multiaxial loading that causes mixed mode long crack growth



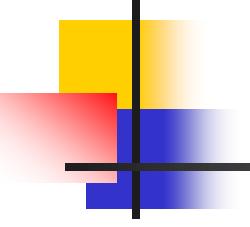
Notched Shaft Loading



Torsion Loading

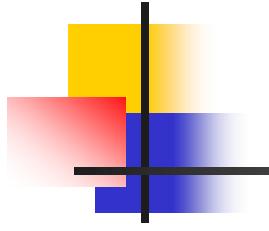


Out-of-phase shear loading is needed to produce nonproportional stressing

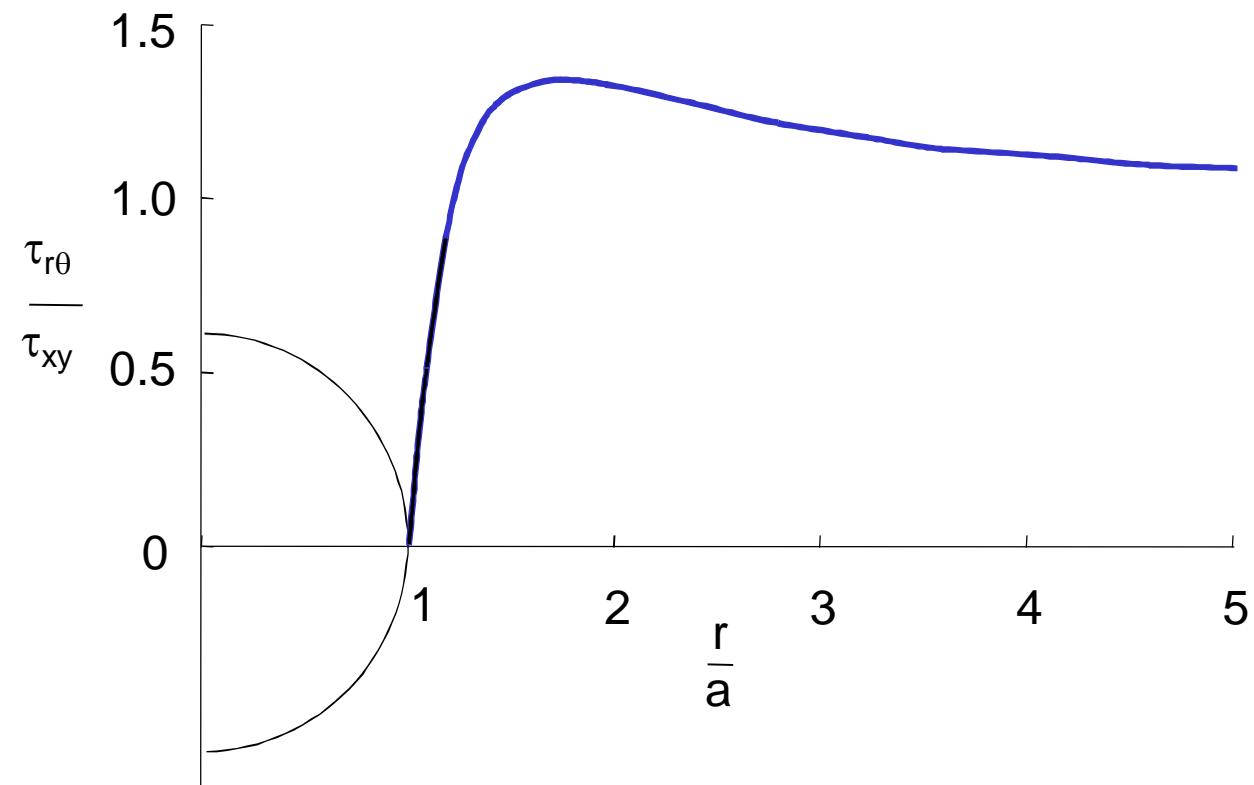
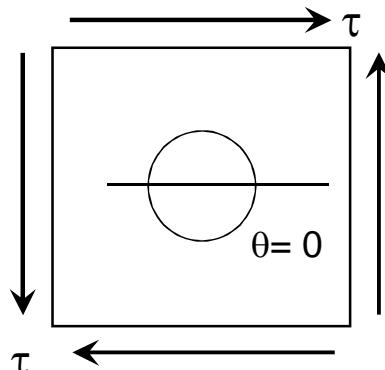


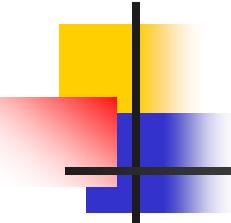
Multiaxial Fatigue Problems

- Uniaxial loading that produces multiaxial stresses around stress concentrators
- Multiaxial loading that produces uniaxial stresses around stress concentrators
- Multiaxial loading that produces multiaxial stresses around stress concentrators
- **Multiaxial loading that causes mixed mode long crack growth**

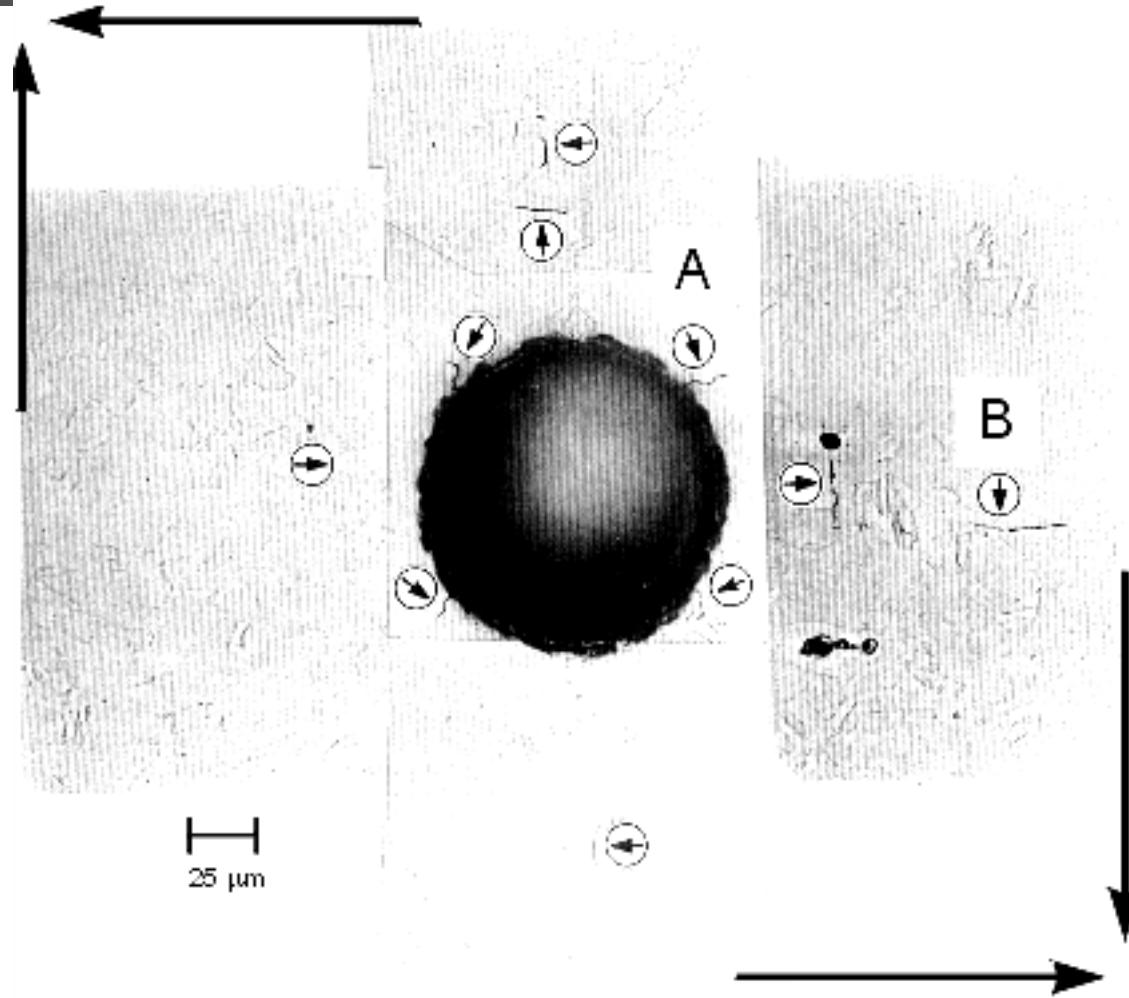


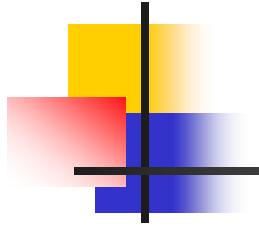
Shear Stresses Around Hole $\theta = 0$



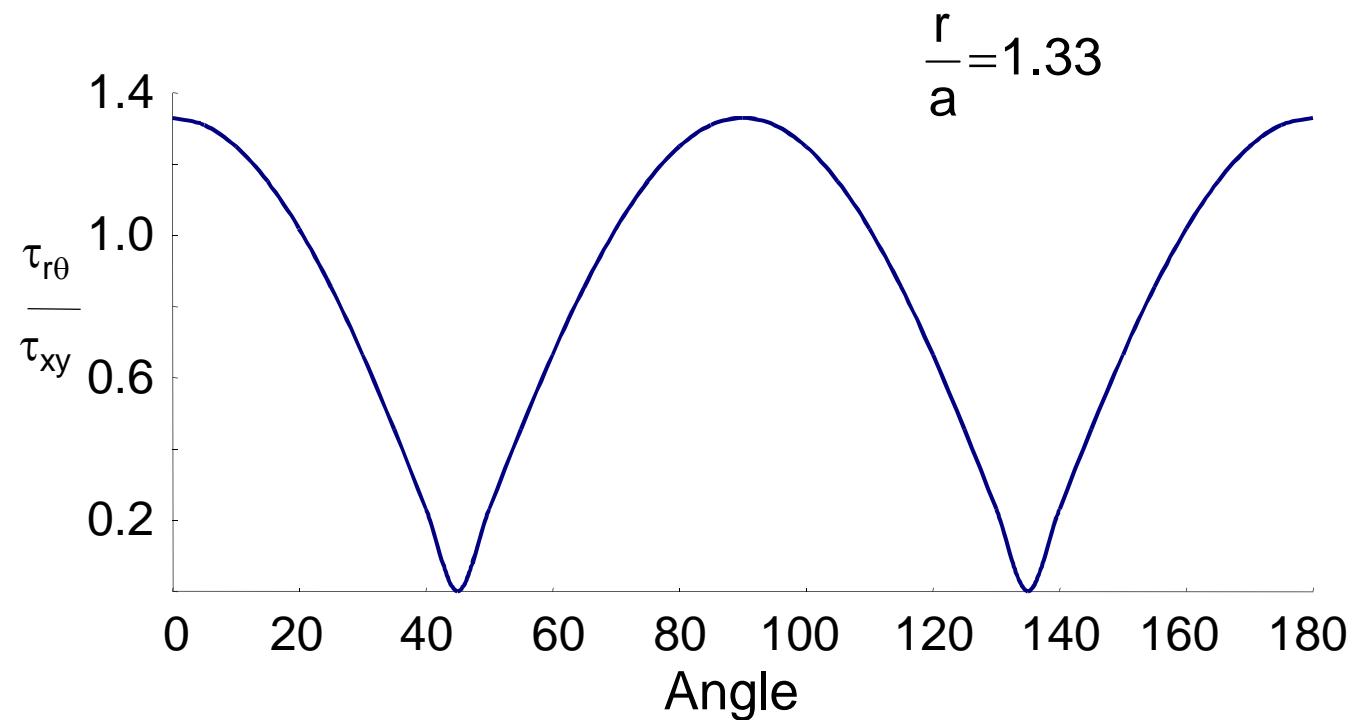
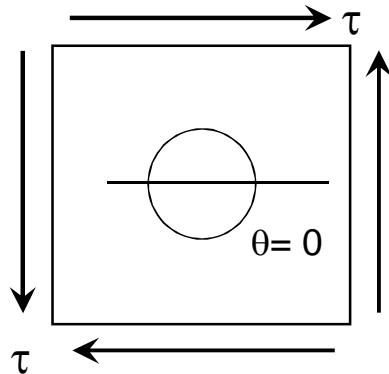


Torsion Experiments

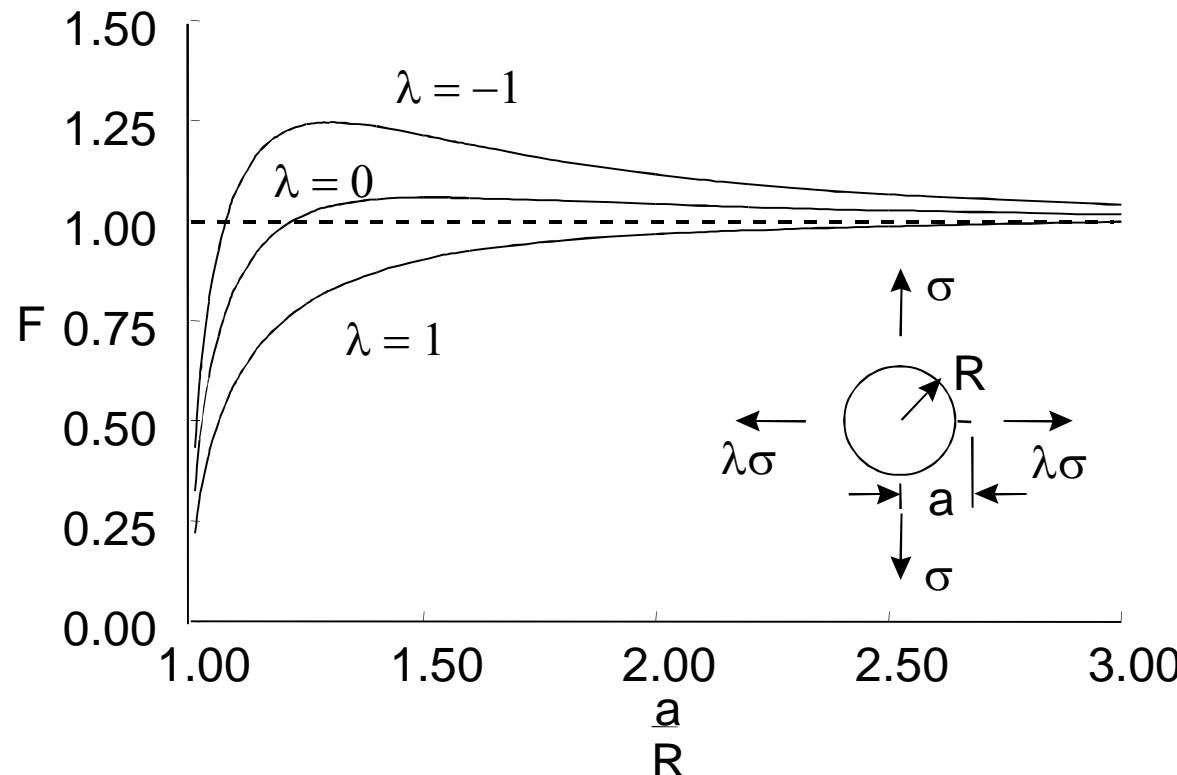




Shear Stresses Around Hole $r = 1.33$

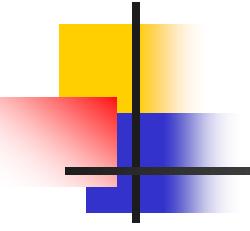


Stress Intensity Factors

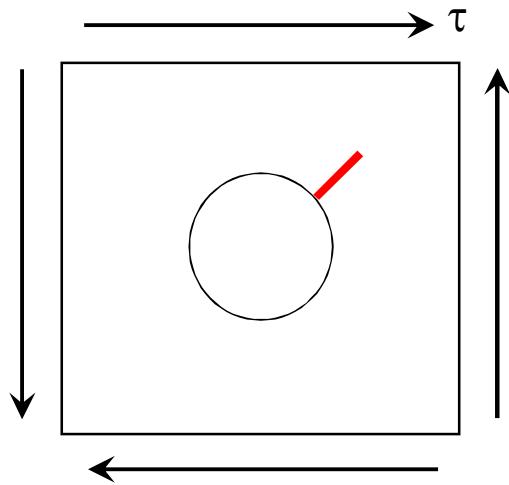


$$\frac{da}{dN} = C (\Delta K_{eq})^m$$

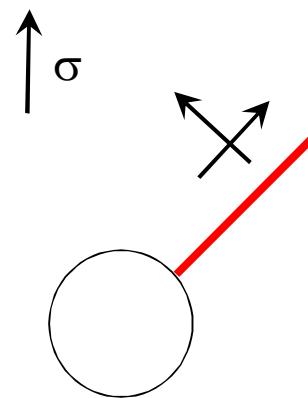
$$K_I = F \Delta\sigma \sqrt{\pi a}$$



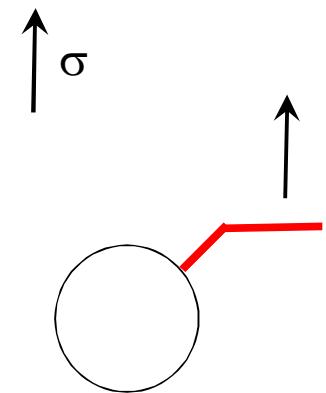
Cracks from Holes



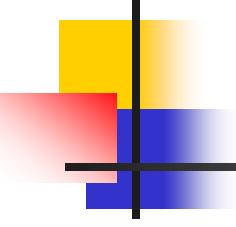
Crack nucleates
in shear



Mixed mode
growth?

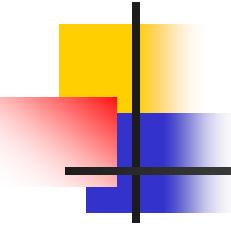


Tensile mode
growth?



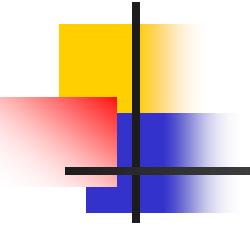
Summary

Cracks nucleate in a uniaxial stress field and then grow in a mixed tensile/shear stress field



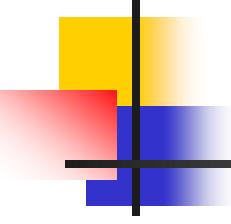
Outline

- Stresses around holes
- Crack Nucleation
 - Stress Based Models
 - Strain Based Models
- Crack Growth



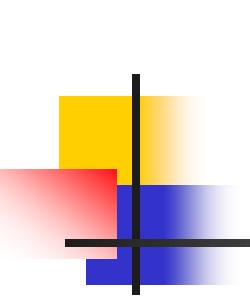
Fatigue Mechanisms Summary

- Fatigue cracks nucleate in shear
- Fatigue cracks grow in either shear or tension depending on material and state of stress

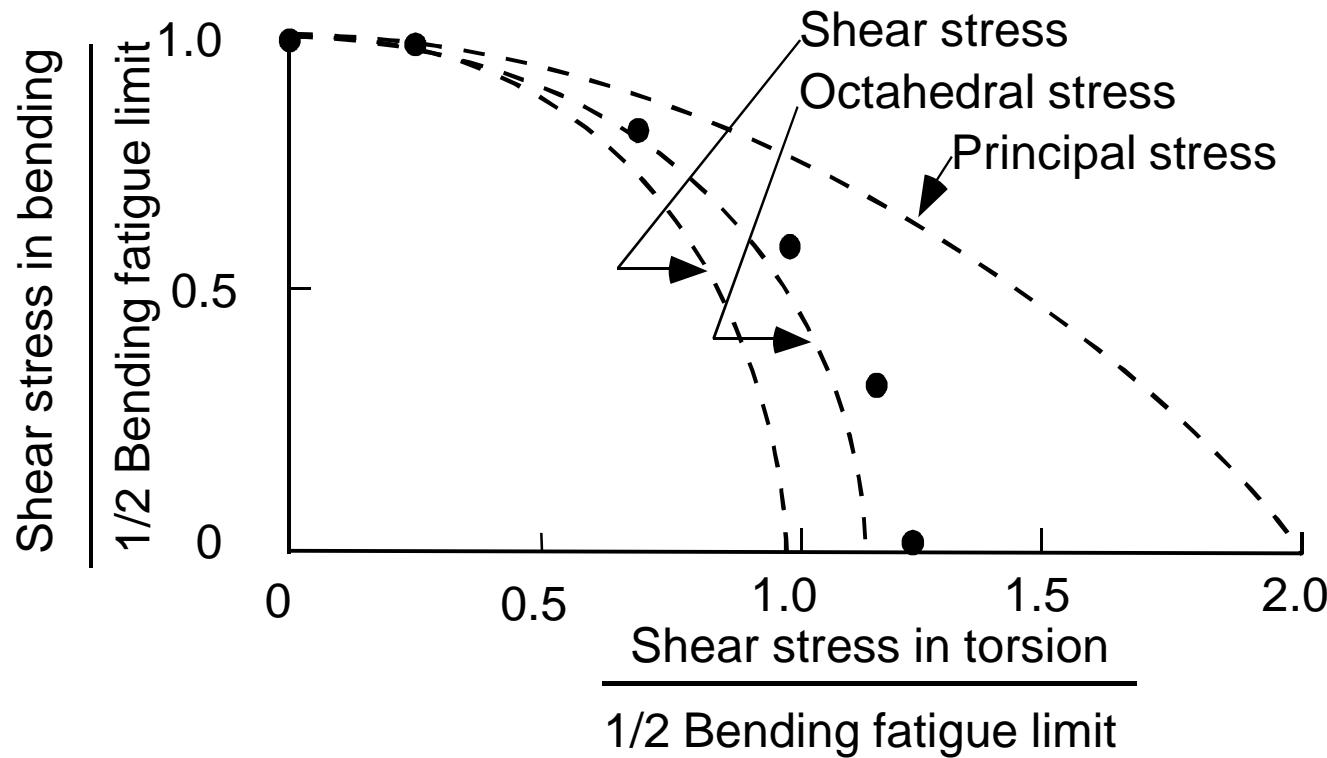


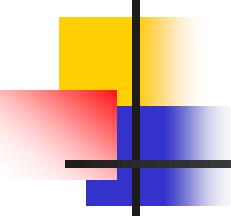
Stress Based Models

- Sines
- Findley
- Dang Van



Bending Torsion Correlation

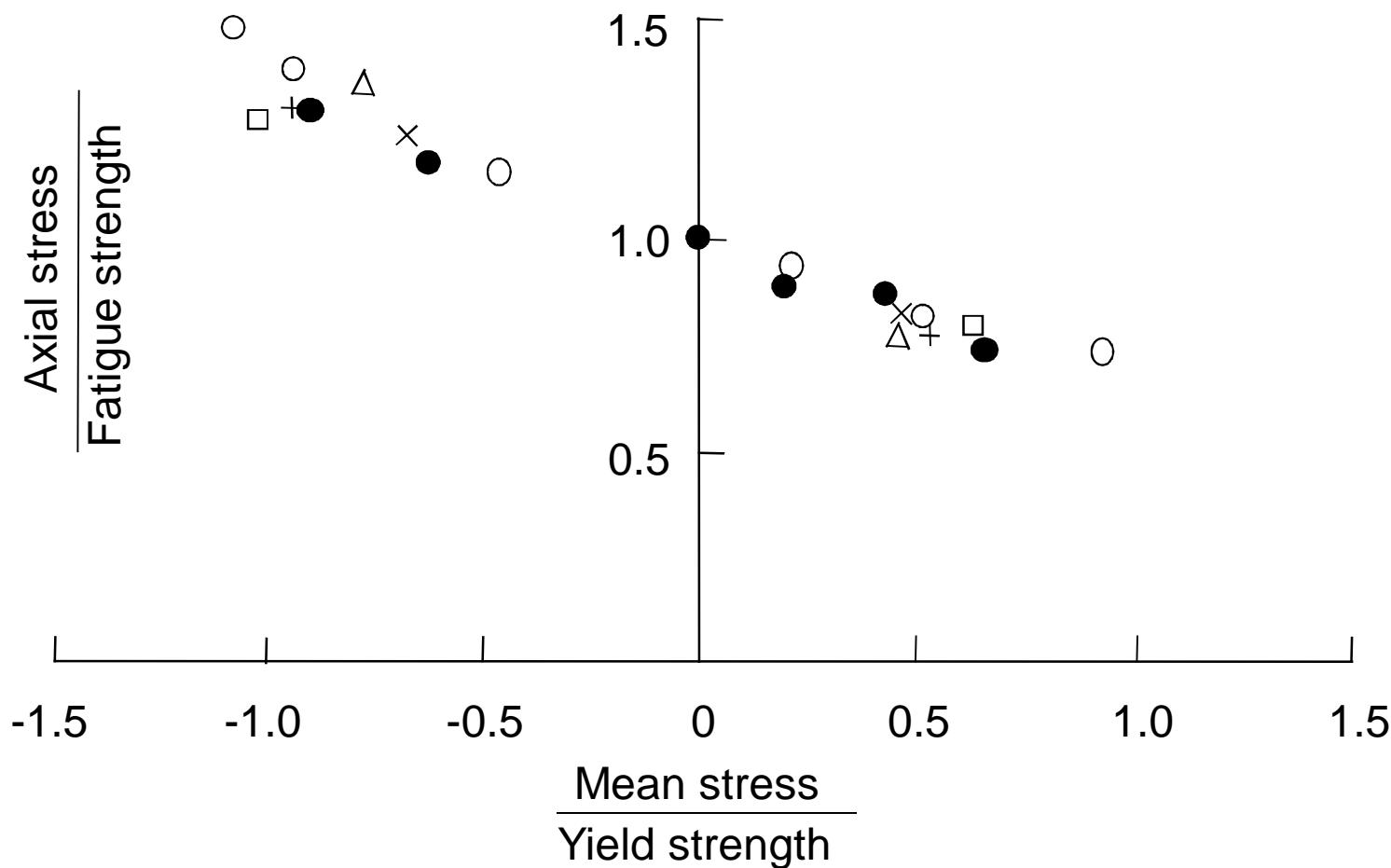




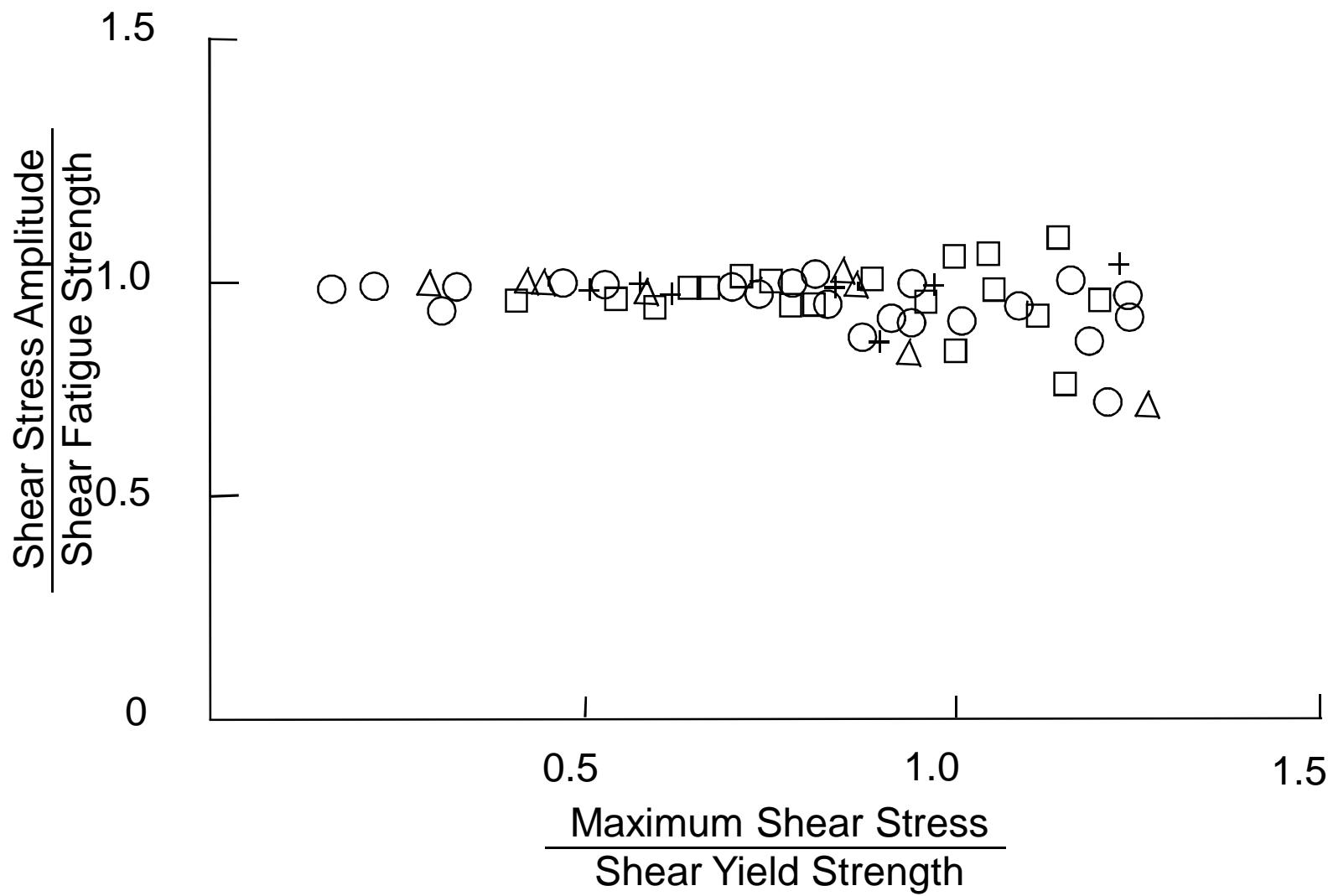
Test Results

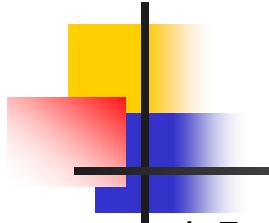
- Cyclic tension with static tension
- Cyclic torsion with static torsion
- Cyclic tension with static torsion
- Cyclic torsion with static tension

Cyclic Tension with Static Tension

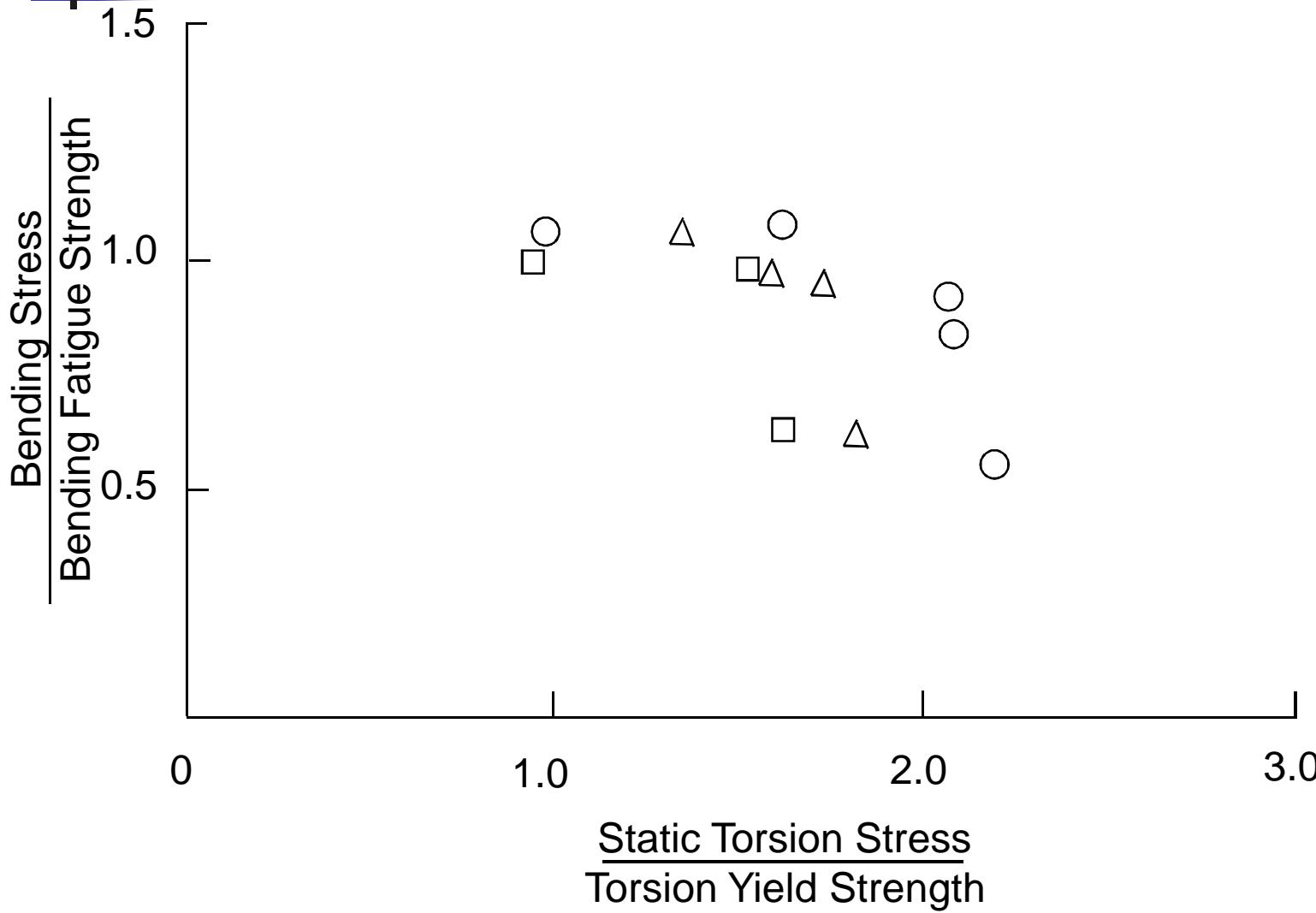


Cyclic Torsion with Static Torsion

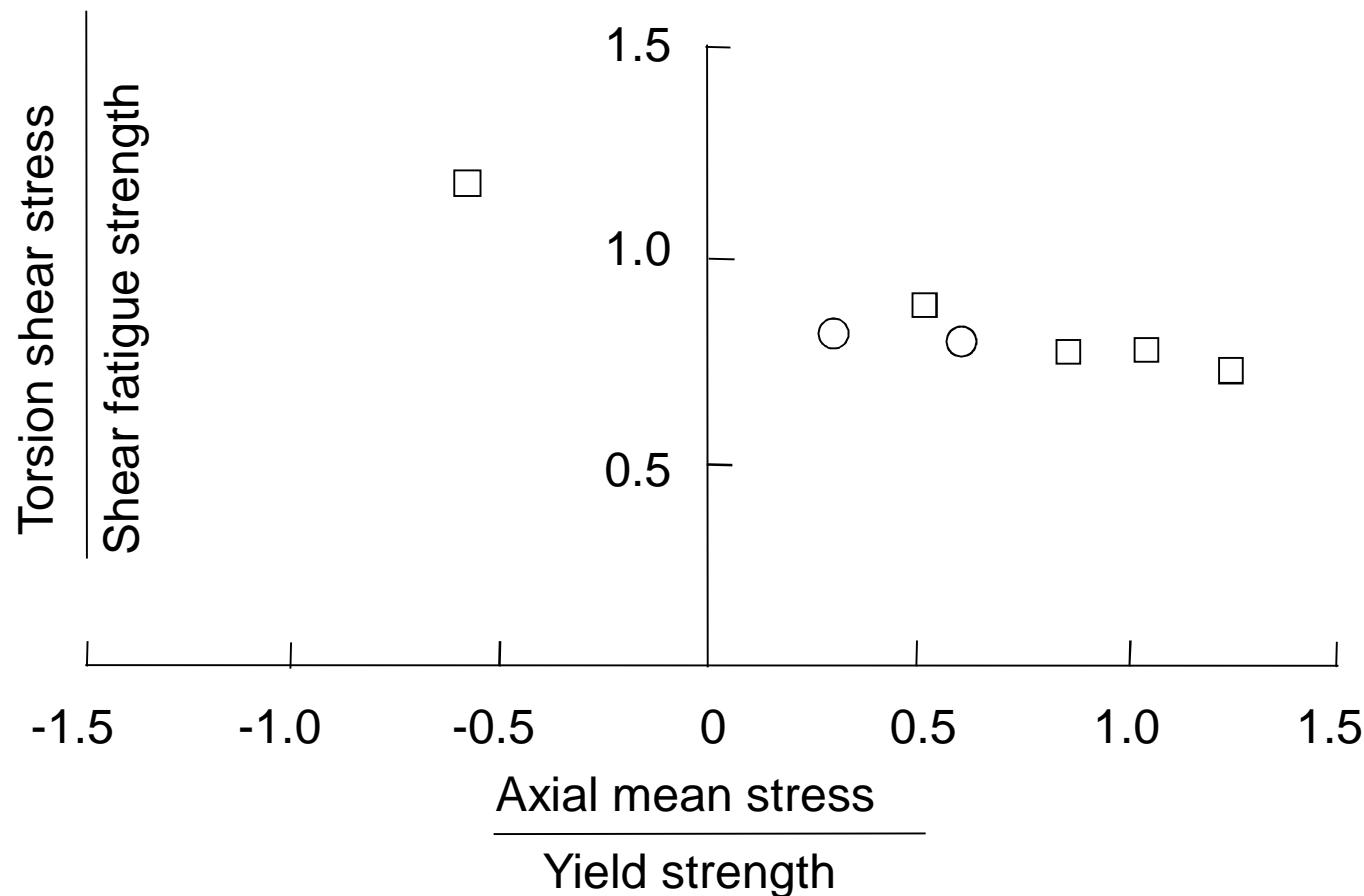


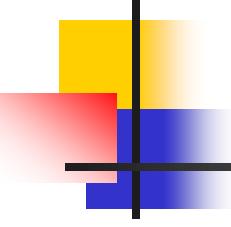


Cyclic Tension with Static Torsion



Cyclic Torsion with Static Tension





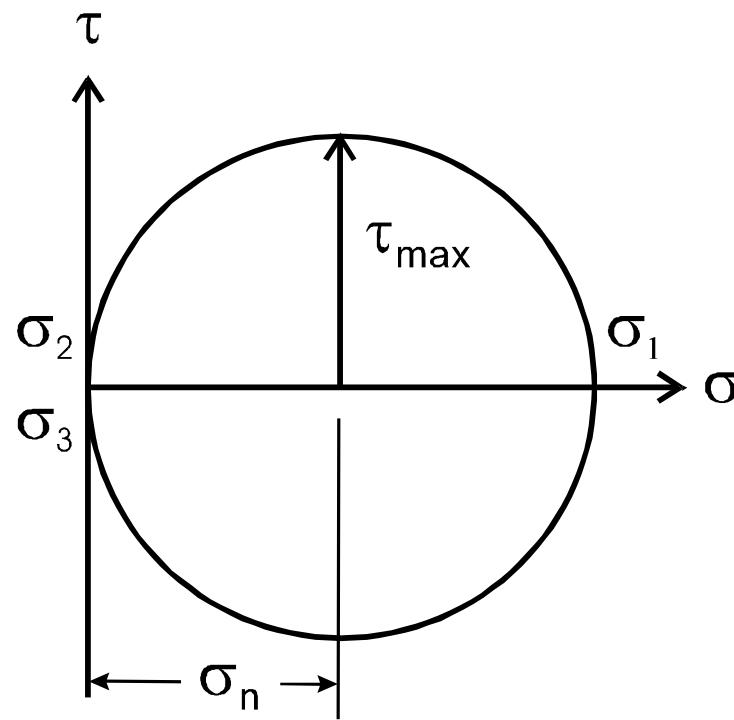
Conclusions

- Tension mean stress affects both tension and torsion
- Torsion mean stress does not affect tension or torsion

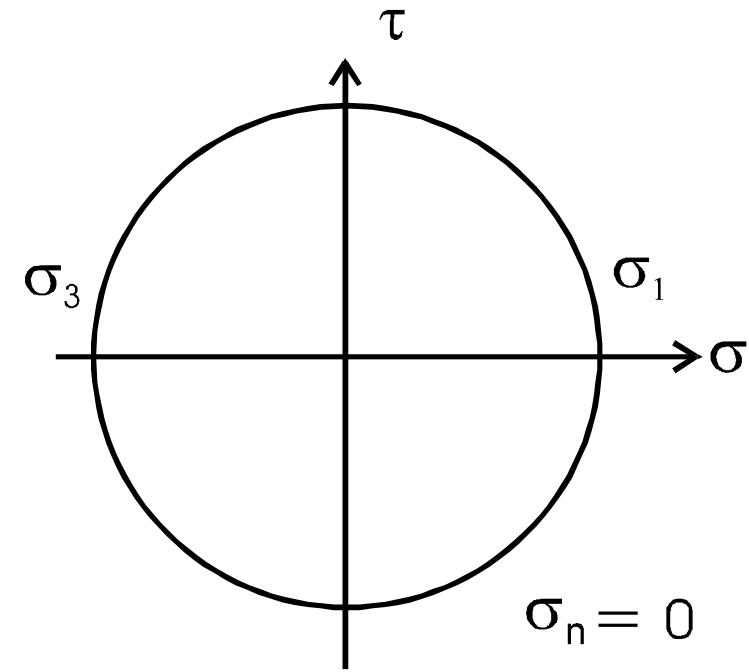
$$\frac{\Delta\tau_{\text{oct}}}{2} + \alpha(3\sigma_h) = \beta$$

$$\frac{1}{6}\sqrt{(\Delta\sigma_x - \Delta\sigma_y)^2 + (\Delta\sigma_x - \Delta\sigma_z)^2 + (\Delta\sigma_y - \Delta\sigma_z)^2 + 6(\Delta\tau_{xy}^2 + \Delta\tau_{xz}^2 + \Delta\tau_{yz}^2)} + \\ \alpha(\sigma_x^{\text{mean}} + \sigma_y^{\text{mean}} + \sigma_z^{\text{mean}}) = \beta$$

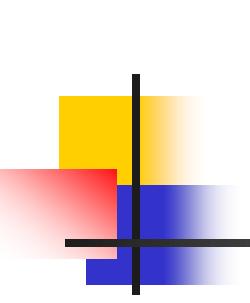
$$\left(\frac{\Delta\tau}{2} + k\sigma_n \right)_{\max} = f$$



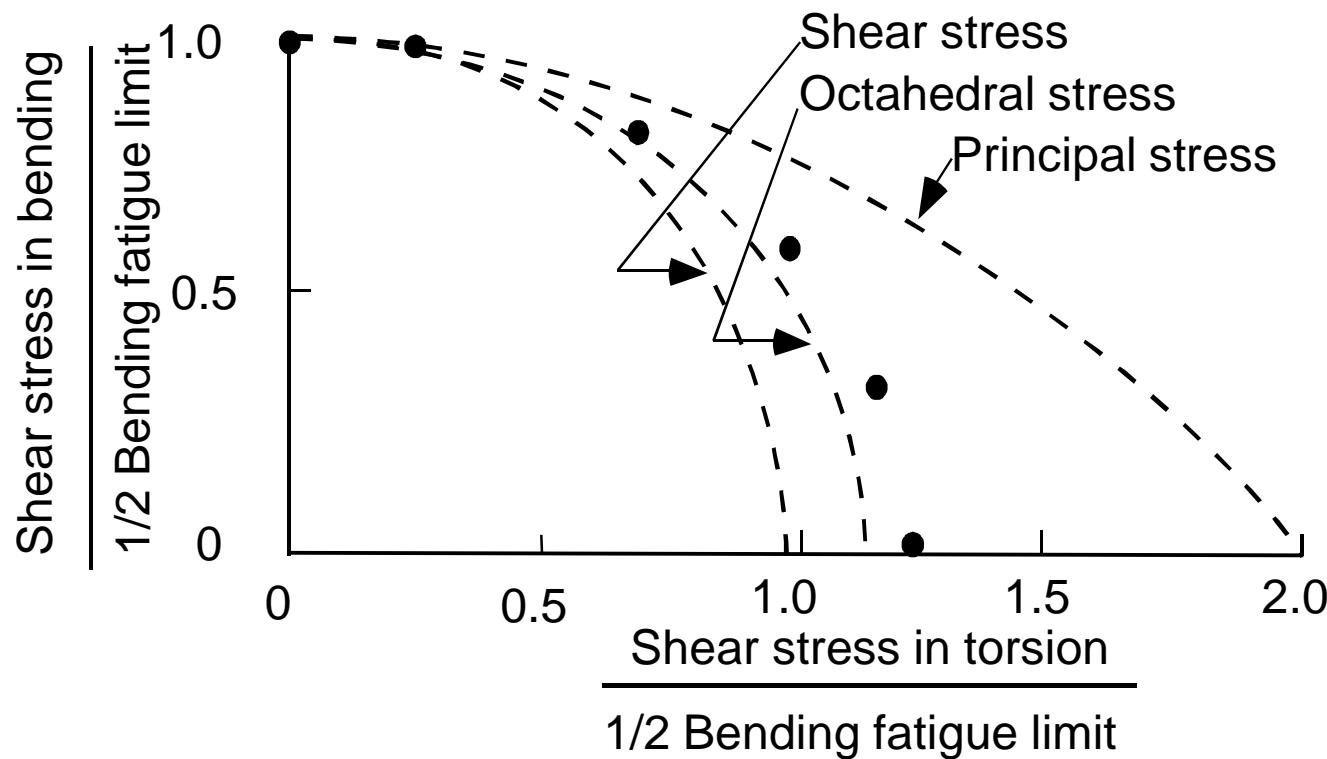
tension

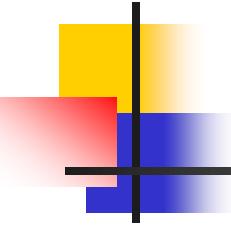


torsion



Bending Torsion Correlation

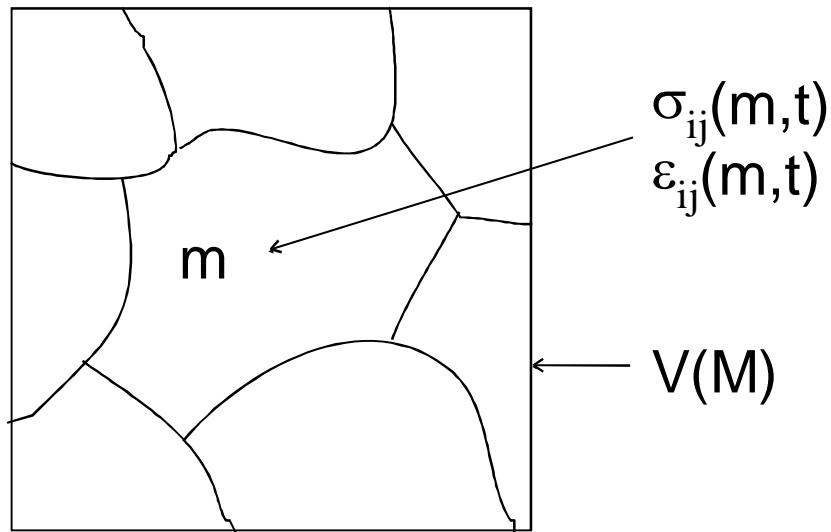


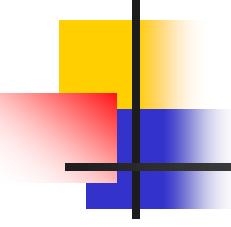


Dang Van

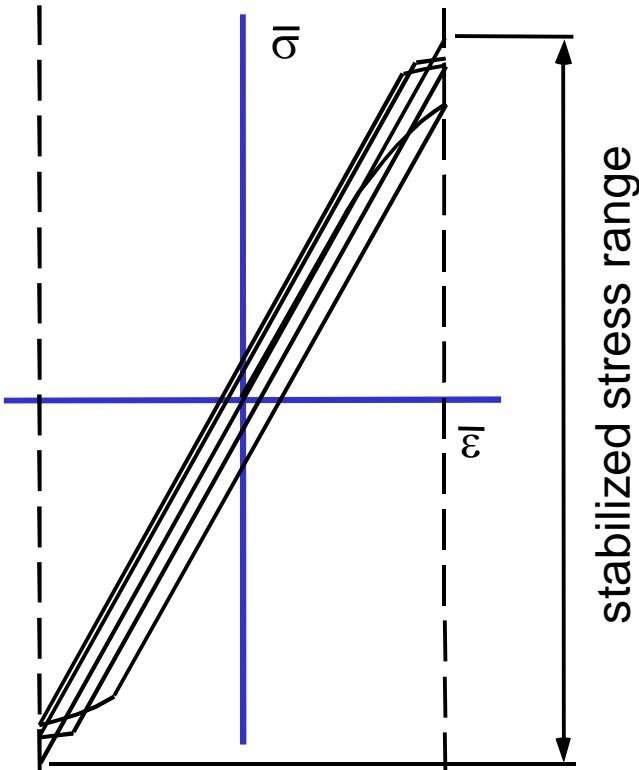
$$\tau(t) + a\sigma_h(t) = b$$

$$\Sigma_{ij}(M,t) \quad E_{ij}(M,t)$$





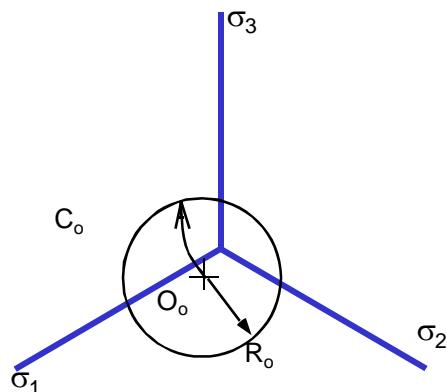
Isotropic Hardening



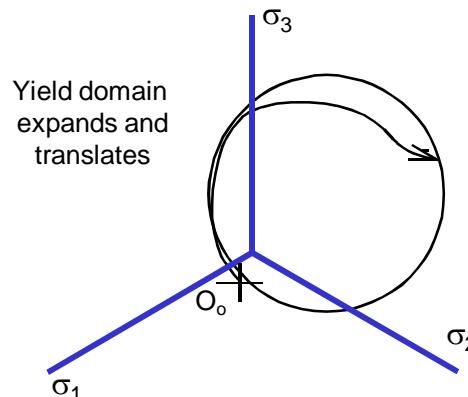
Failure occurs when
the stress range is
not elastic

Multiaxial Kinematic and Isotropic

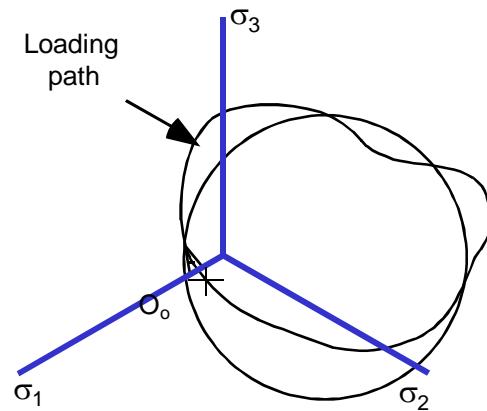
a)



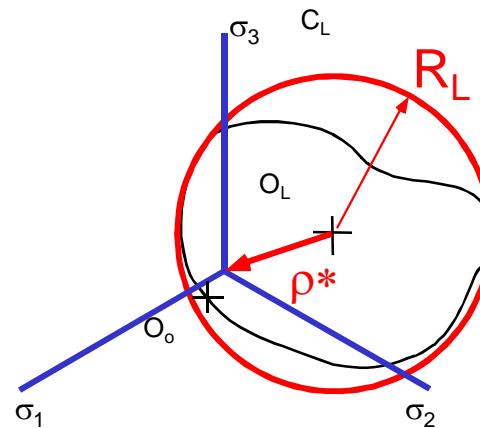
b)



c)

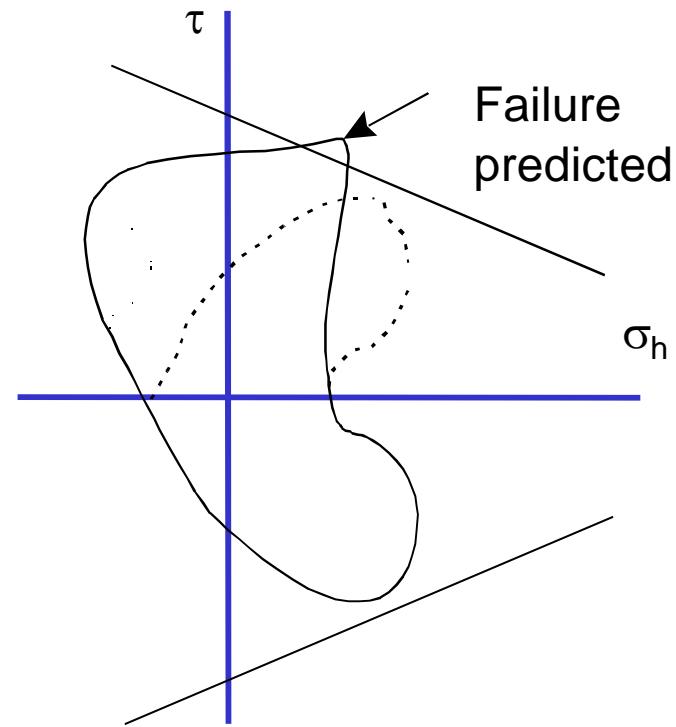
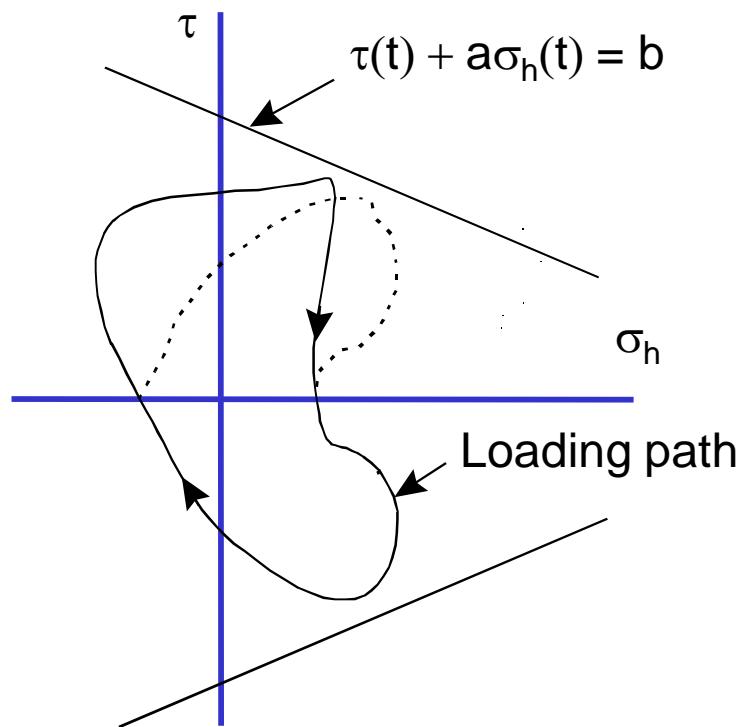


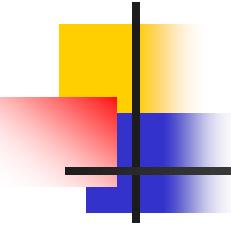
d)



ρ^* stabilized residual stress

Dang Van (continued)





Stress Based Models Summary

Sines:

$$\frac{\Delta\tau_{oct}}{2} + \alpha(3\sigma_h) = \beta$$

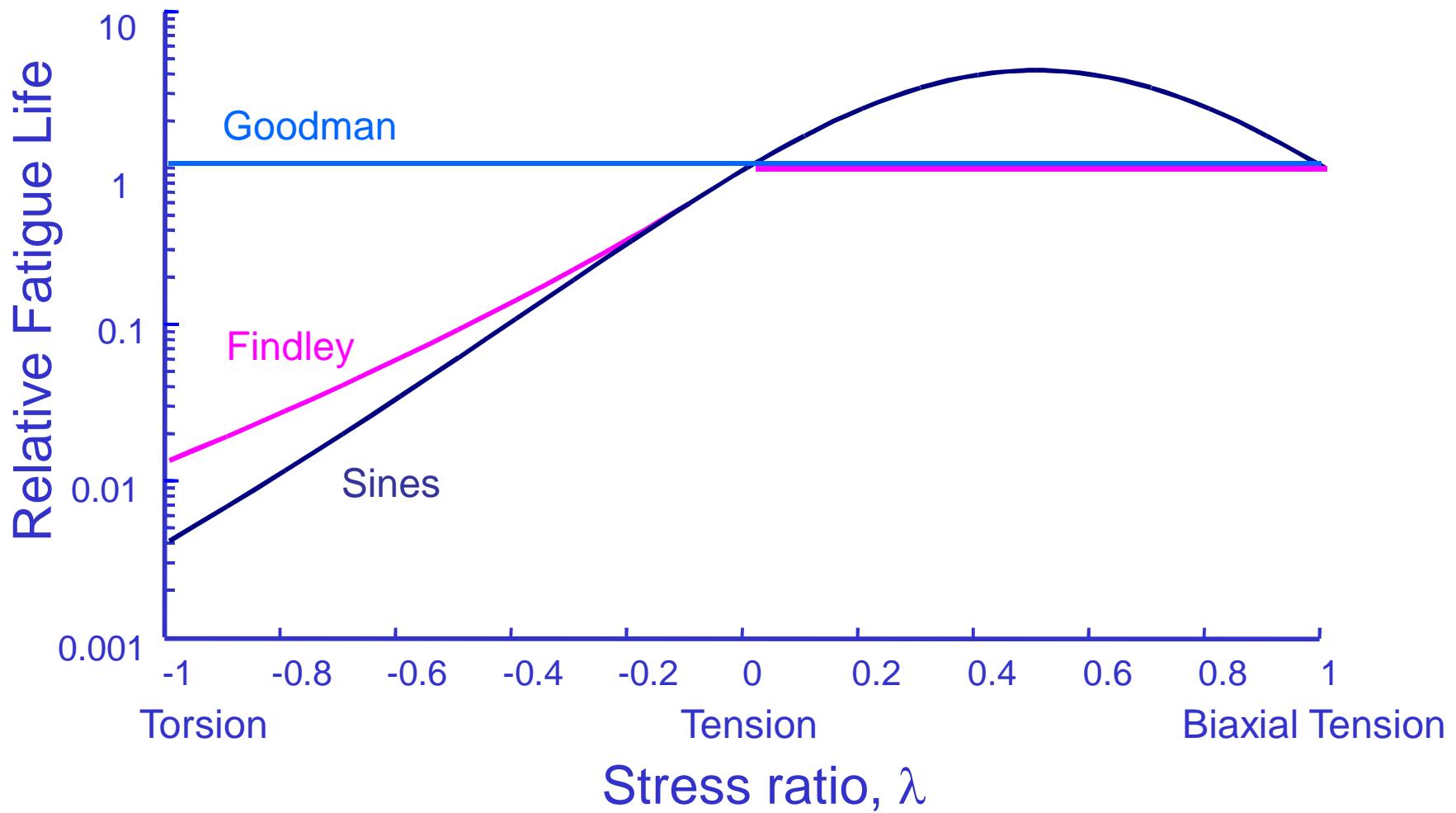
Findley:

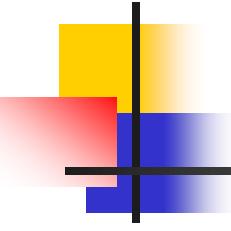
$$\left(\frac{\Delta\tau}{2} + k\sigma_n \right)_{max} = f$$

Dang Van:

$$\tau(t) + a\sigma_h(t) = b$$

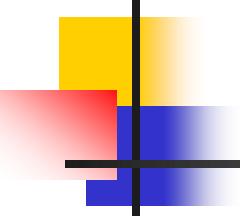
Model Comparison $R = -1$





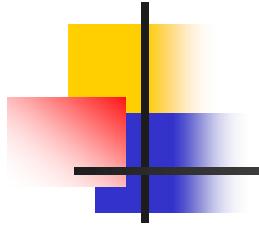
Outline

- Stresses around holes
- Crack Nucleation
 - Stress Based Models
 - Strain Based Models
- Crack Growth

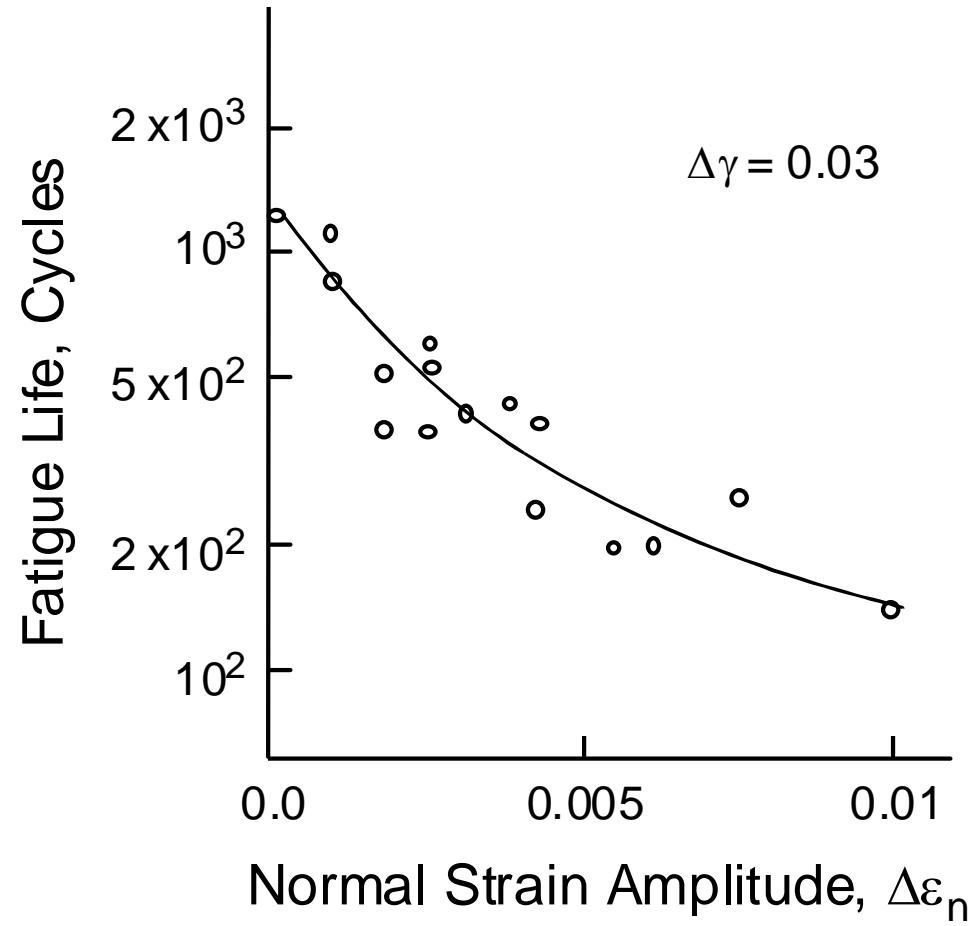


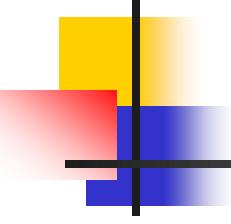
Strain Based Models

- Brown and Miller
- Fatemi and Socie
- Smith Watson and Topper

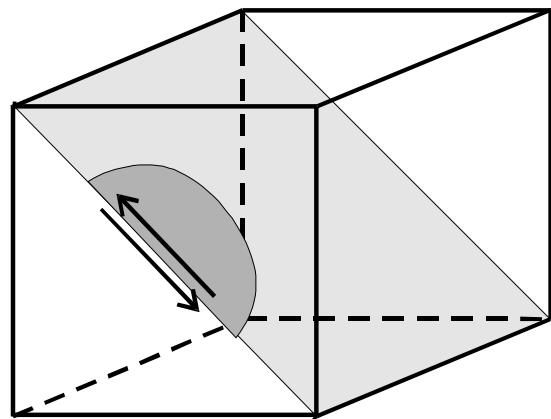


Brown and Miller



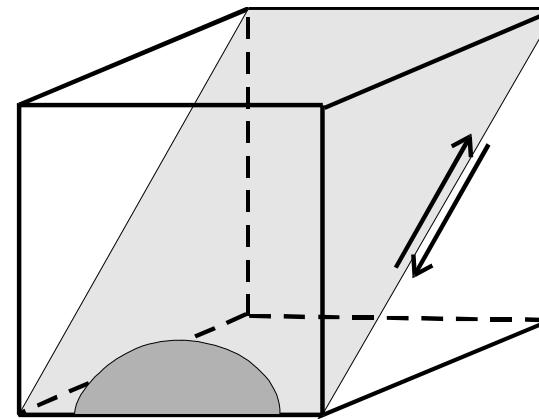


Case A and B



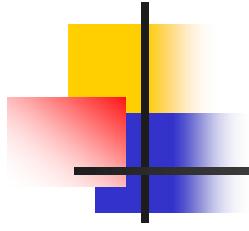
Case A

Growth along the surface

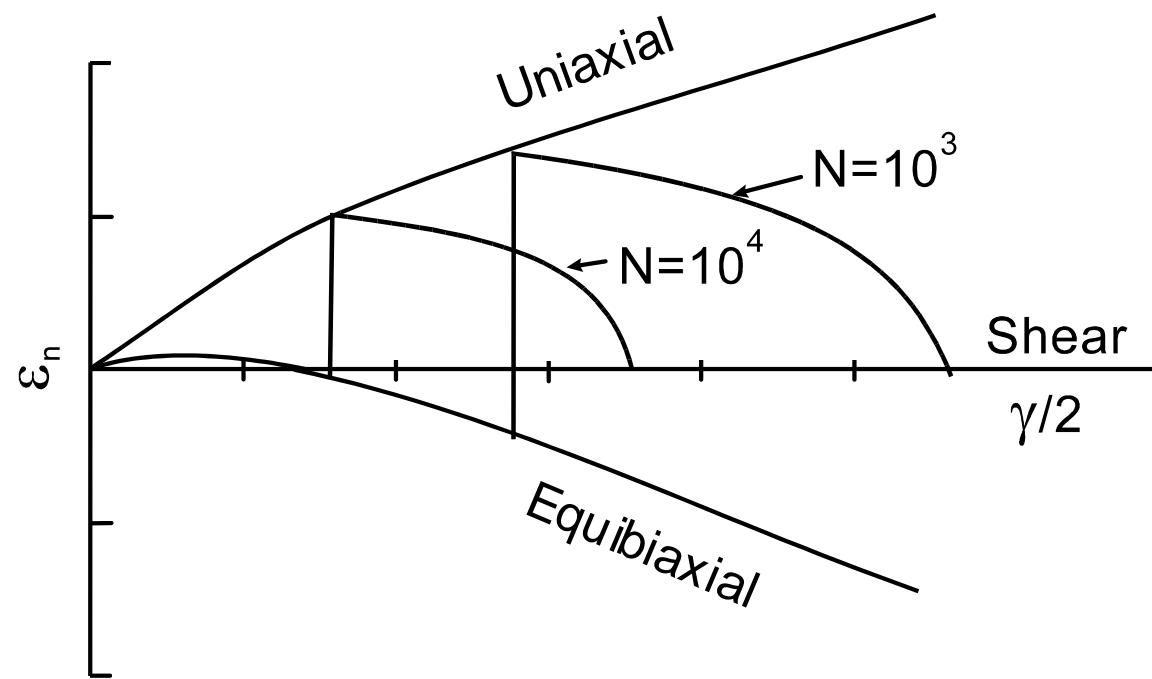


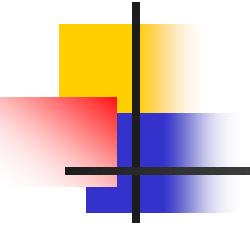
Case B

Growth into the surface



Brown and Miller (continued)



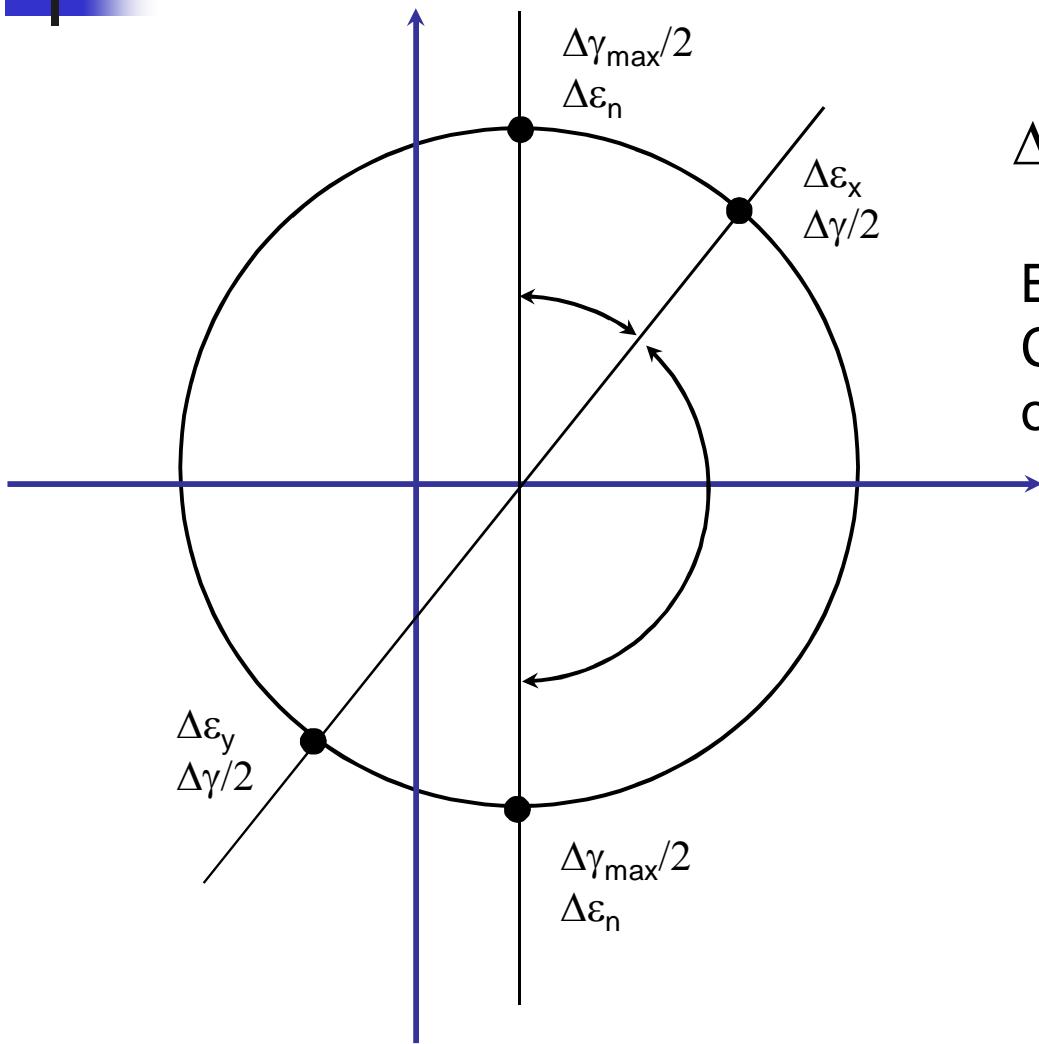


Brown and Miller (continued)

$$\Delta\hat{\gamma} = \left(\Delta\gamma_{\max}^{\alpha} + S\Delta\varepsilon_n^{\alpha} \right)^{\frac{1}{\alpha}}$$

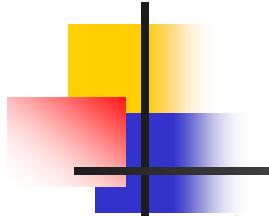
$$\frac{\Delta\gamma_{\max}}{2} + S\Delta\varepsilon_n = A \frac{\sigma_f' - 2\sigma_{n,\text{mean}}}{E} (2N_f)^b + B\varepsilon_f'(2N_f)^c$$

Mohr's Circle

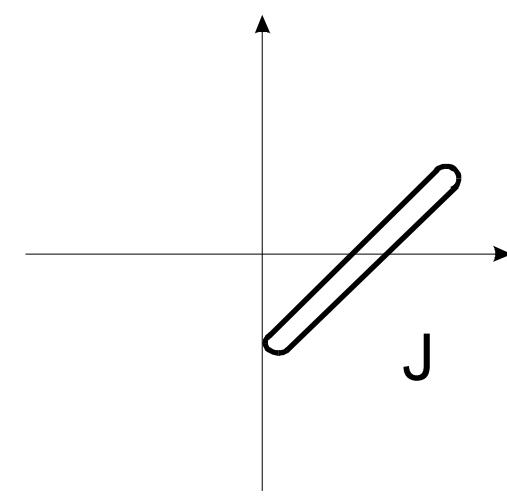
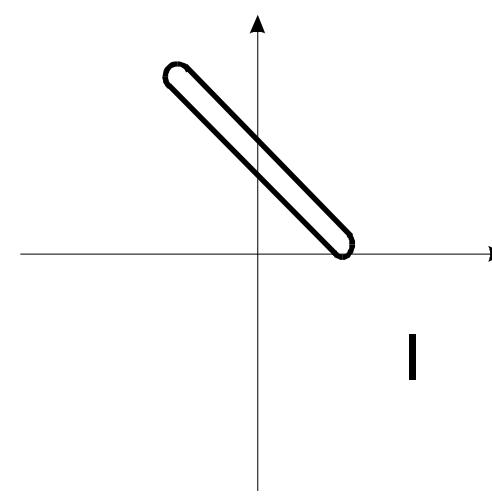
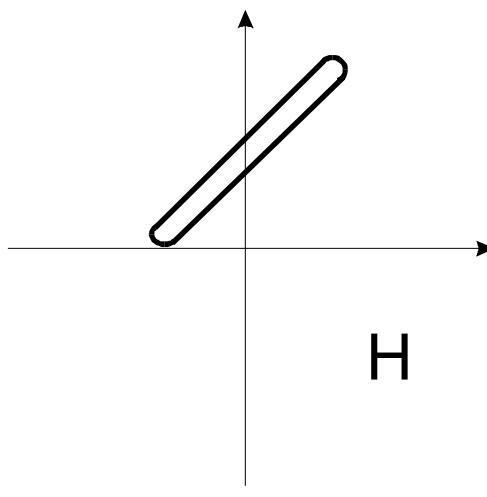
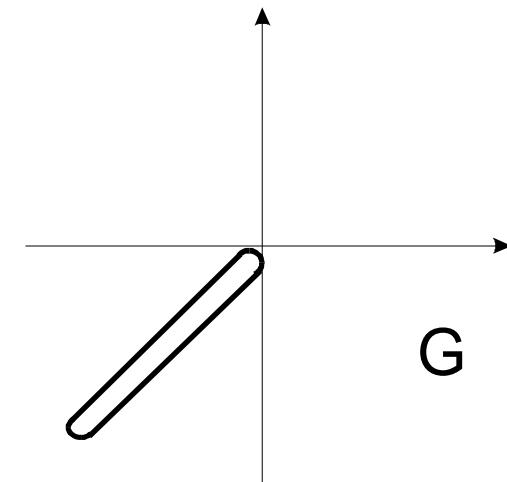
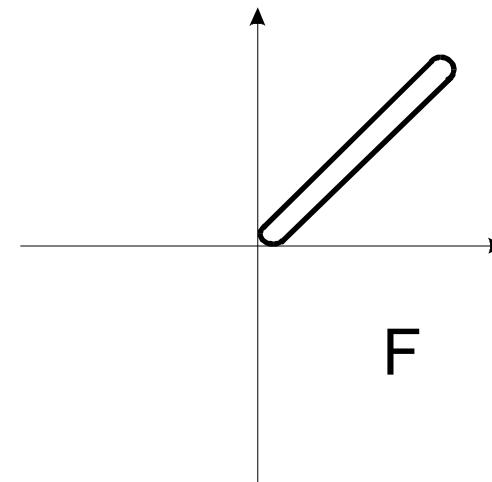
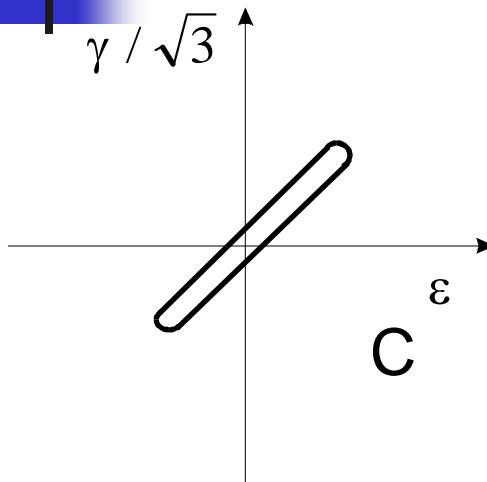


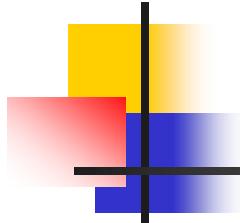
$$\Delta\hat{\gamma} = \left(\Delta\gamma_{\max}^{\alpha} + S\Delta\varepsilon_n^{\alpha} \right)^{\frac{1}{\alpha}}$$

Brown and Miller -
Cracks should be equally likely
on two planes 90° apart

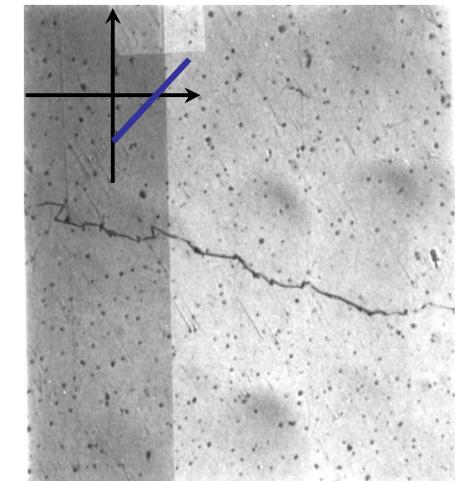
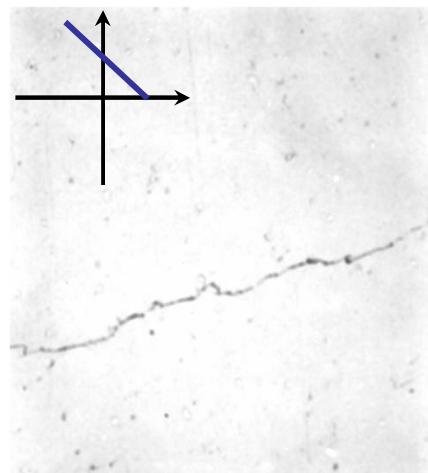
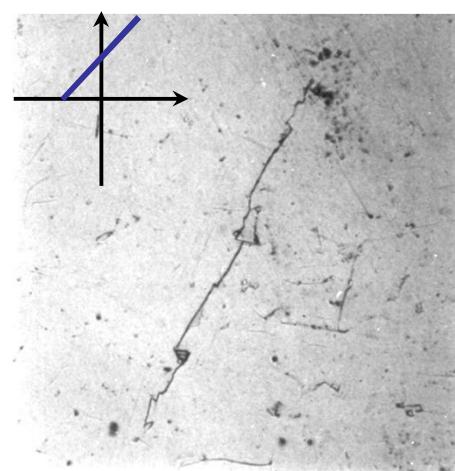
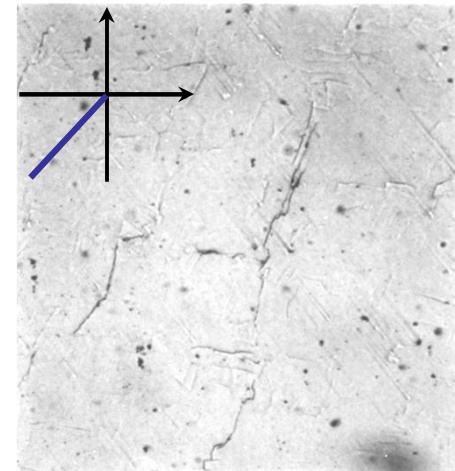
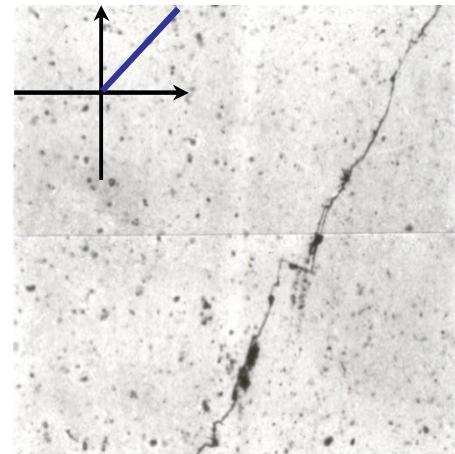
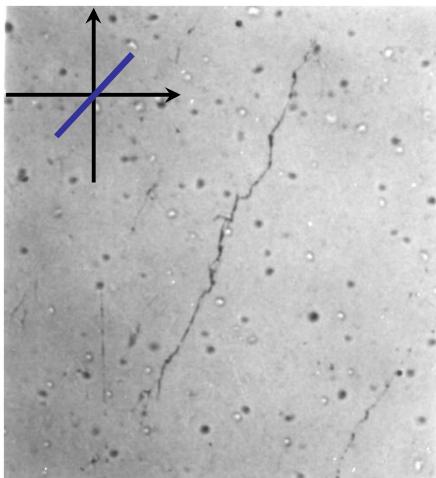


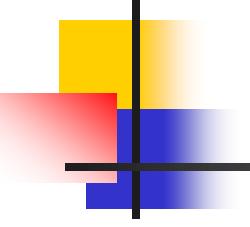
Loading Histories





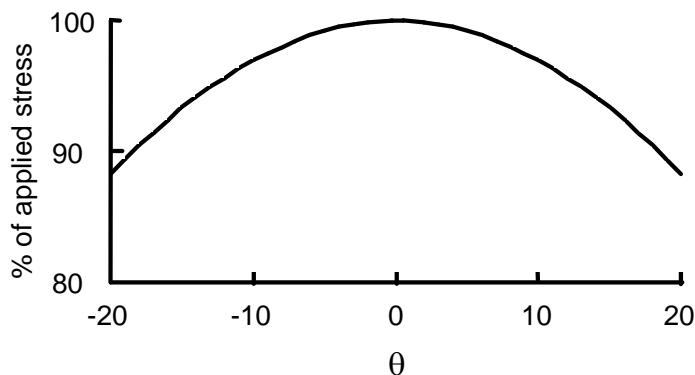
Crack Directions





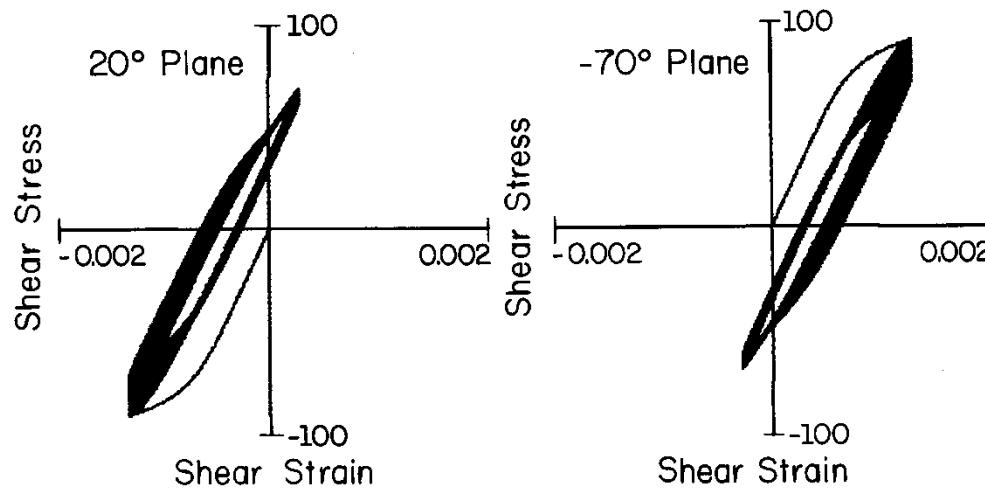
Hypothesis

Fatigue damage is planar in nature
The material finds a critical plane for microcrack growth.

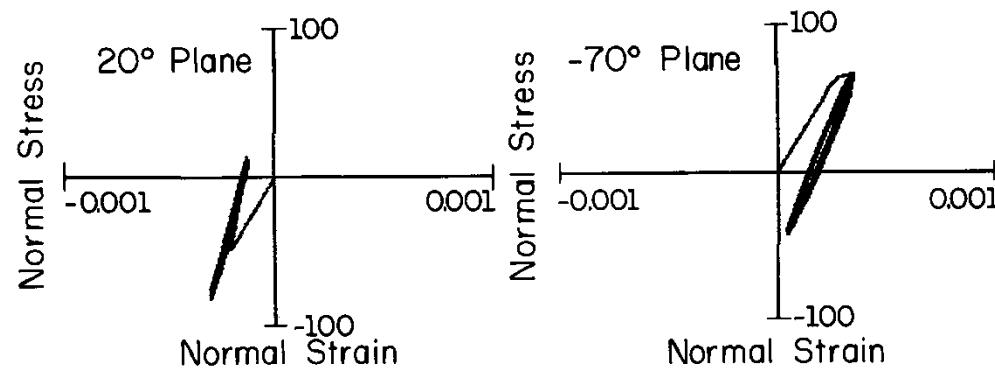


Stresses are nearly the same over a 10° range of angles

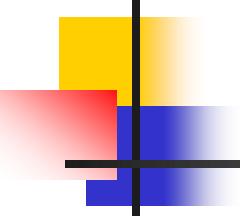
Stresses on the Planes



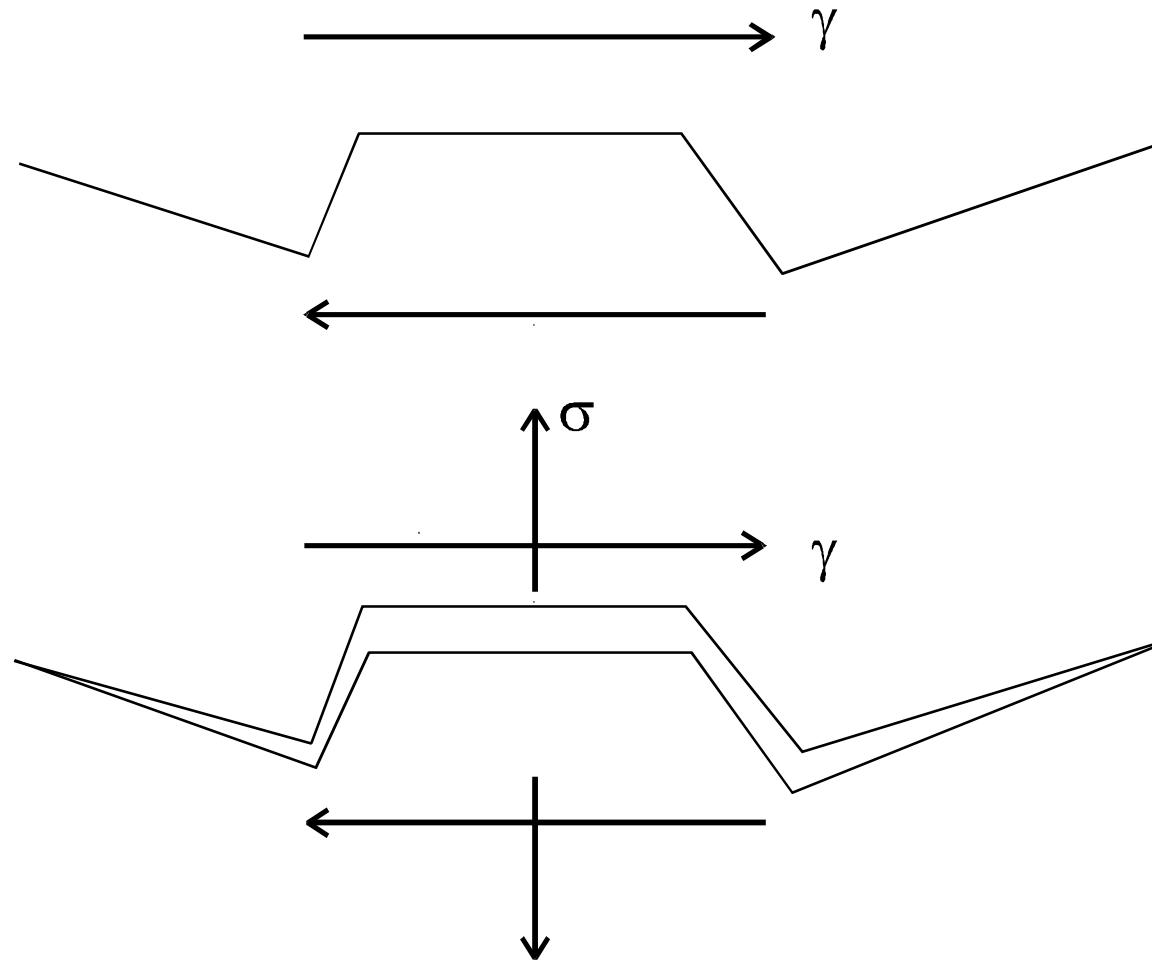
Shear

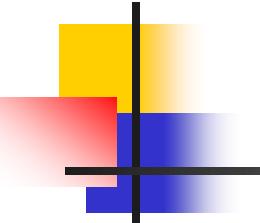


Tension

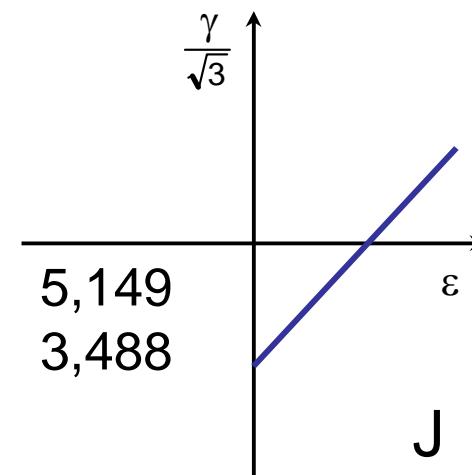
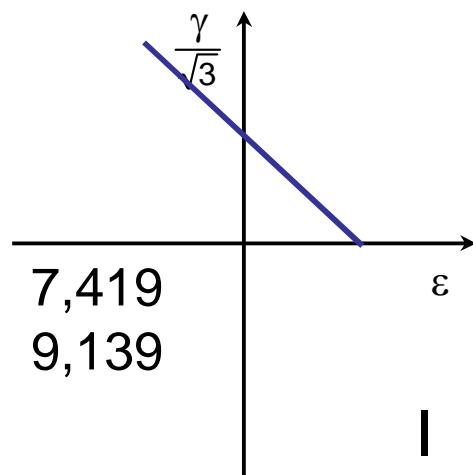
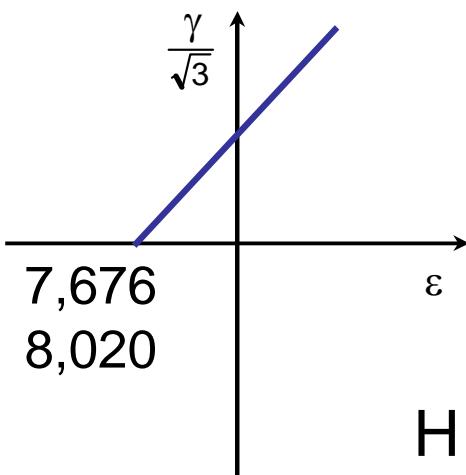
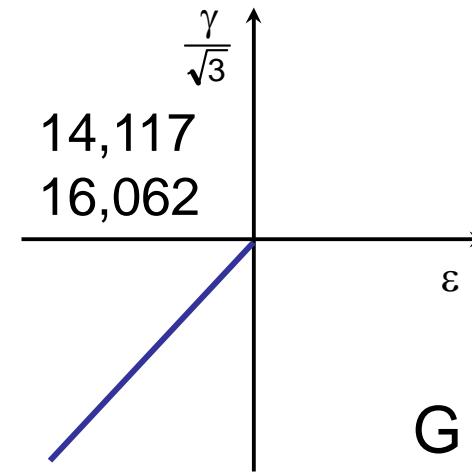
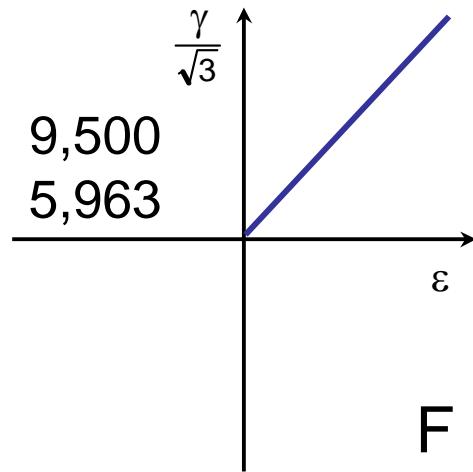
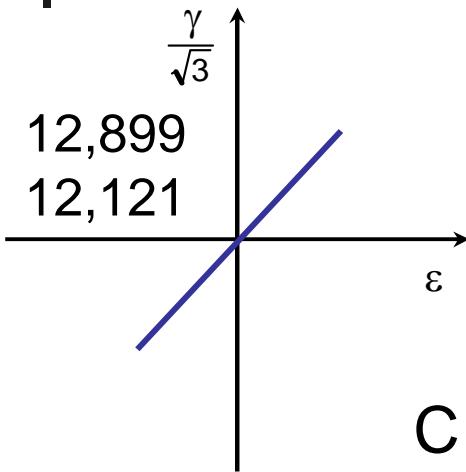


Fatemi and Socie

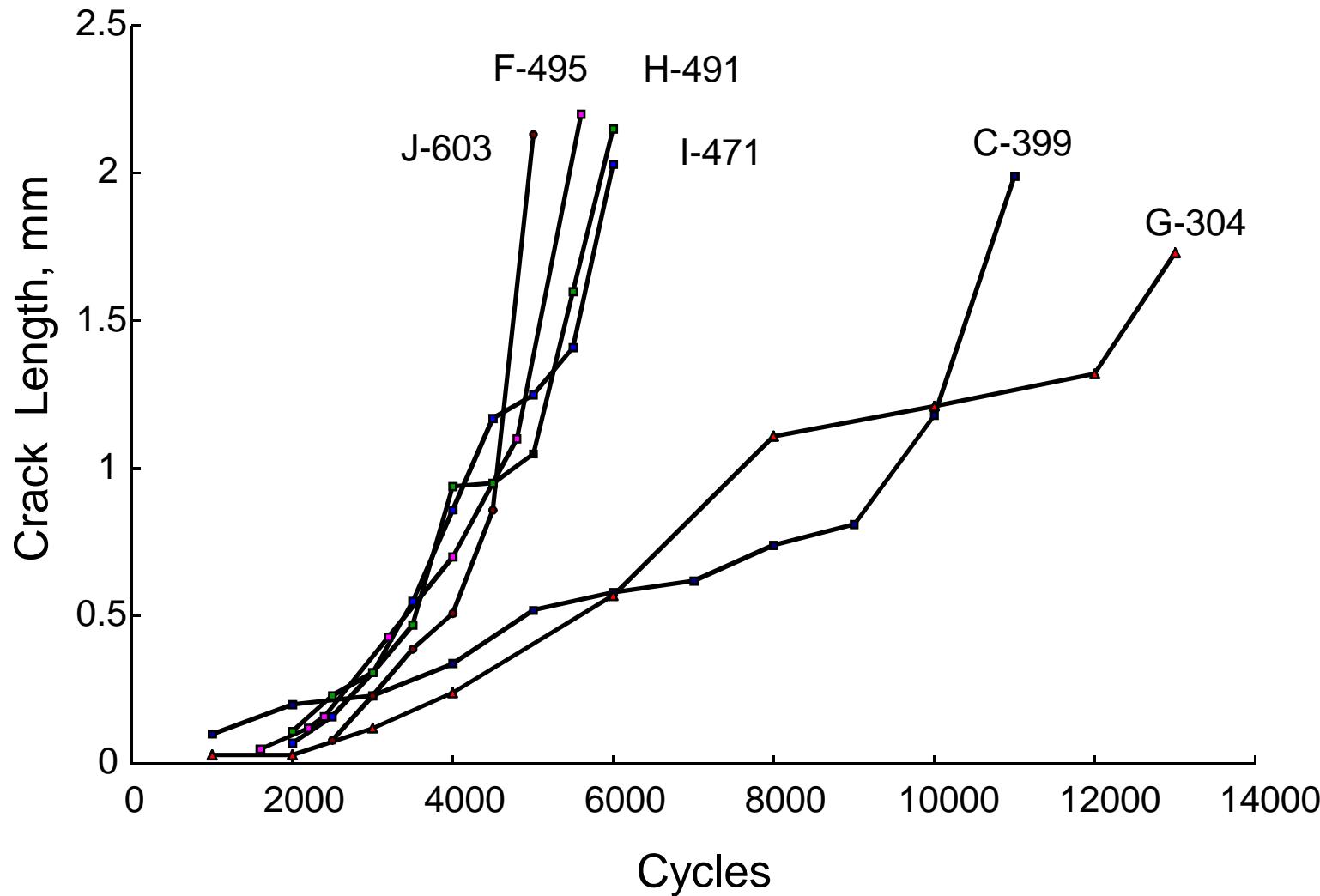


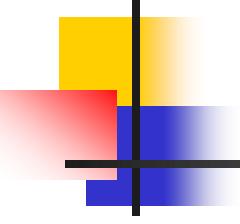


Fatigue Lives



Crack Length Observations

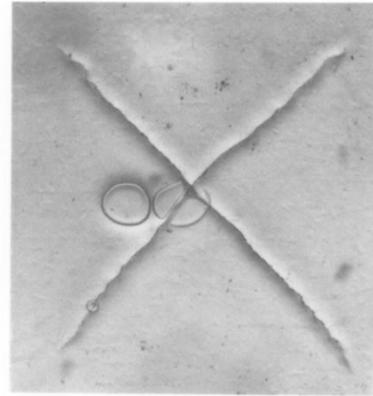
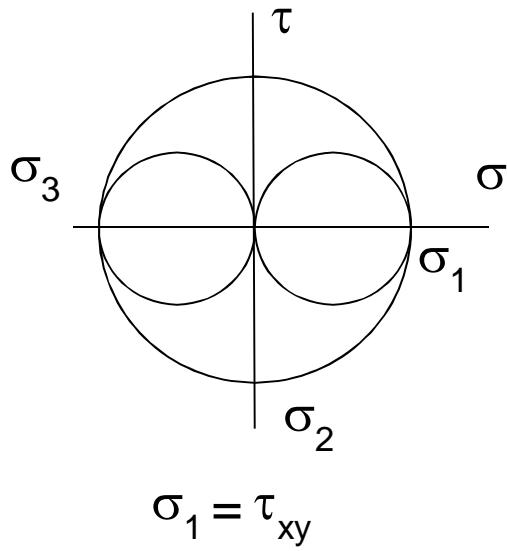




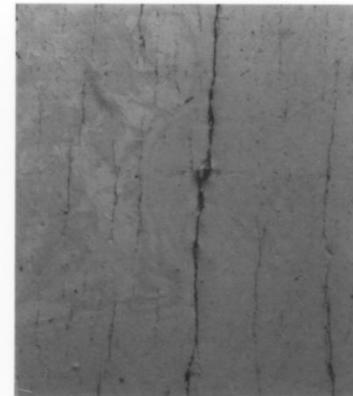
Fatemi and Socie

$$\frac{\Delta\gamma}{2} \left(1 + k \frac{\sigma_{n,max}}{\sigma_y} \right) = \frac{\tau_f}{G} (2N_f)^{bo} + \gamma_f (2N_f)^{co}$$

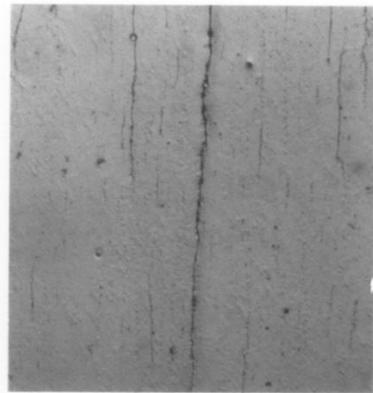
Torsion Tests



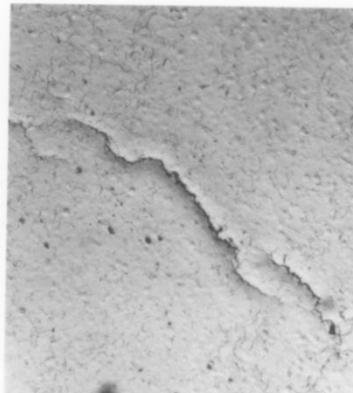
STAINLESS STEEL AISI 304 (SS09)
 $\frac{\Delta\gamma}{2} = 0.35\%$ $N = 1.0 \times 10^6$
 $N_f = 1.1 \times 10^6$



INCONEL 718 (IN13) $N = 1500$
 $\frac{\Delta\gamma}{2} = 1.7\%$ $N_f = 1670$

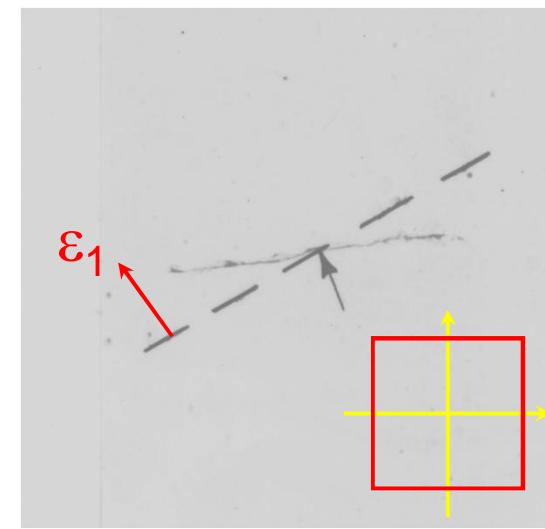
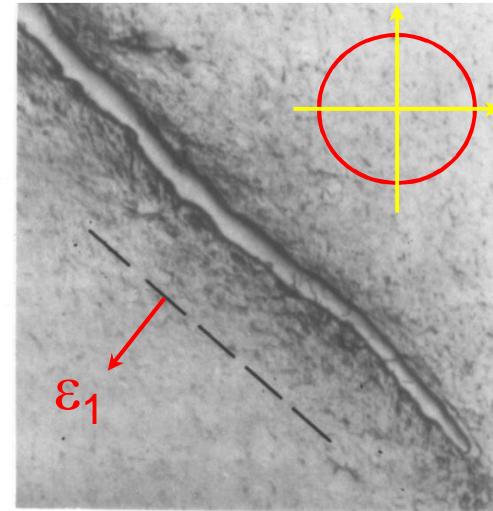
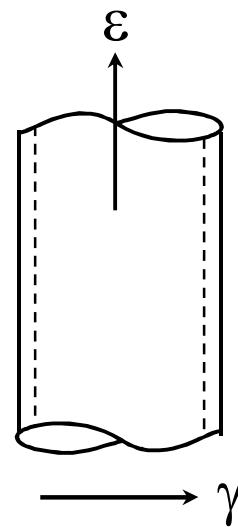
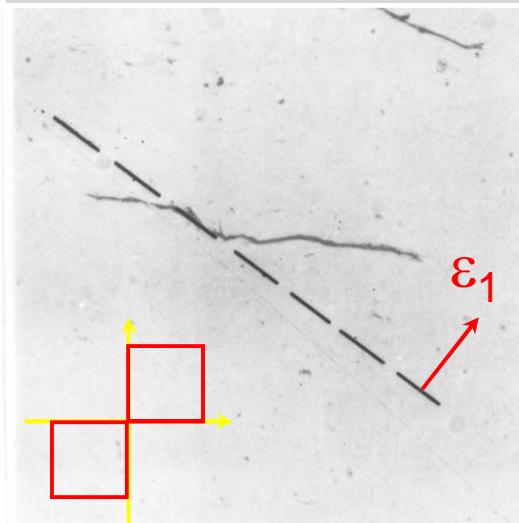
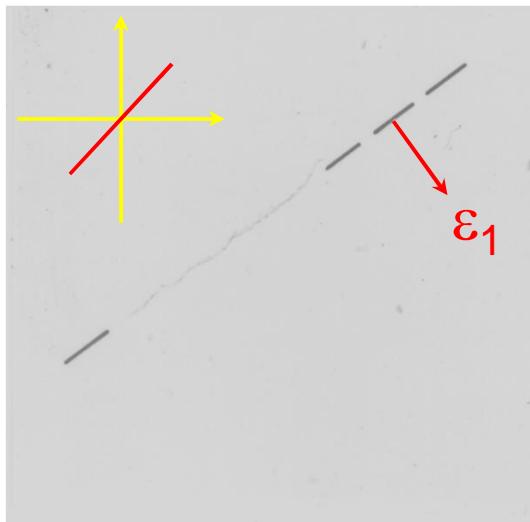


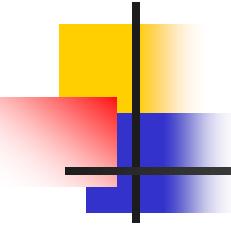
SAE 1045 (4531) $N = 90,000$
 $\frac{\Delta\gamma}{2} = 0.38\%$ $N_f = 93052$



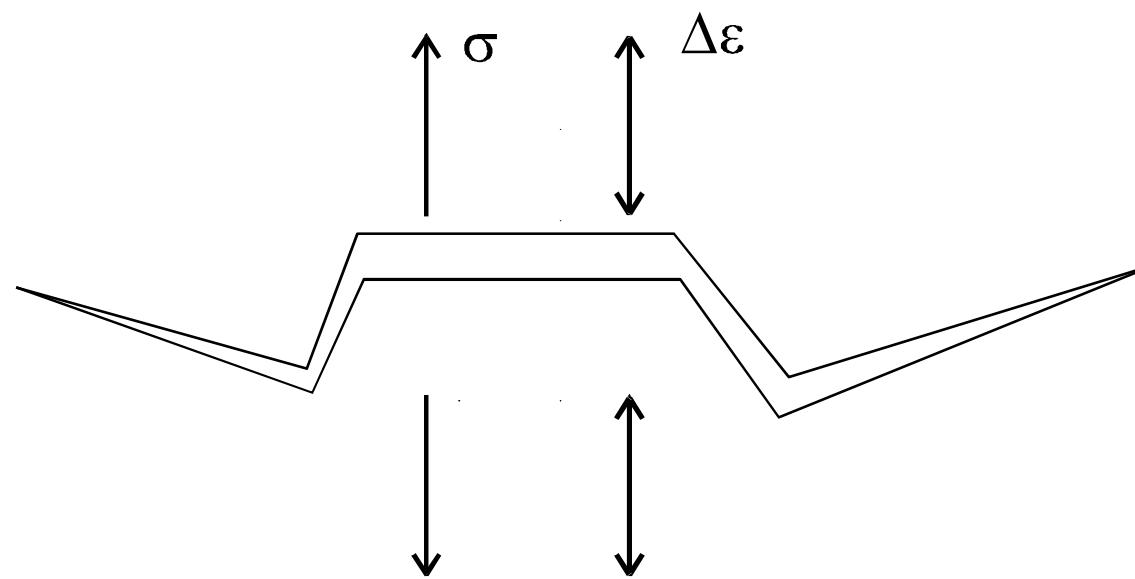
GRAY CAST IRON (NC01) $N = 400$
 $\frac{\Delta\gamma}{2} = .006$ $N_f = 440$

304 Stainless Steel

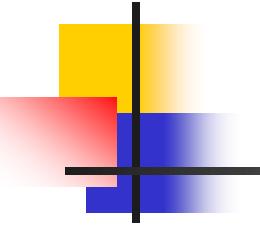




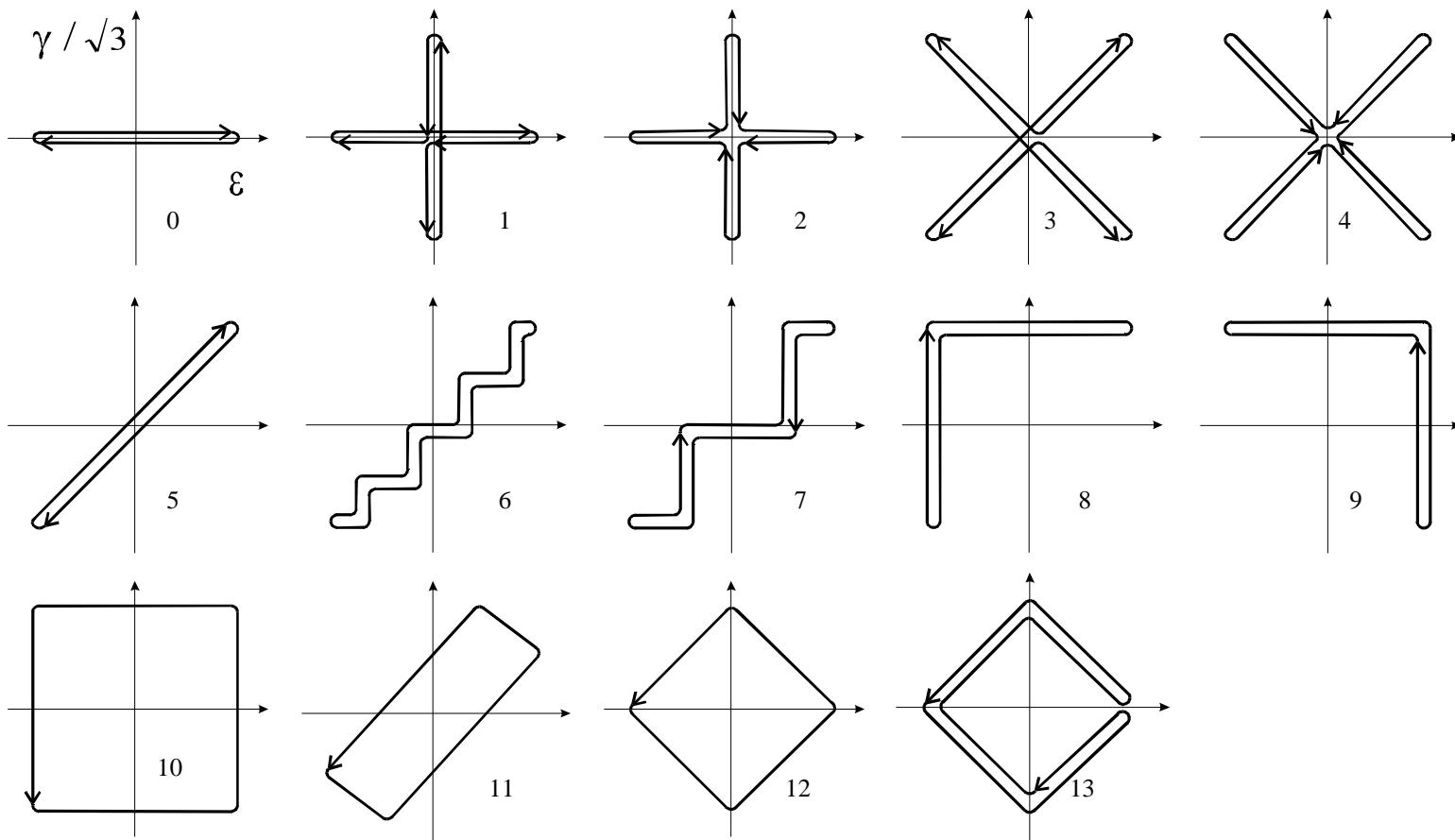
Smith Watson Topper



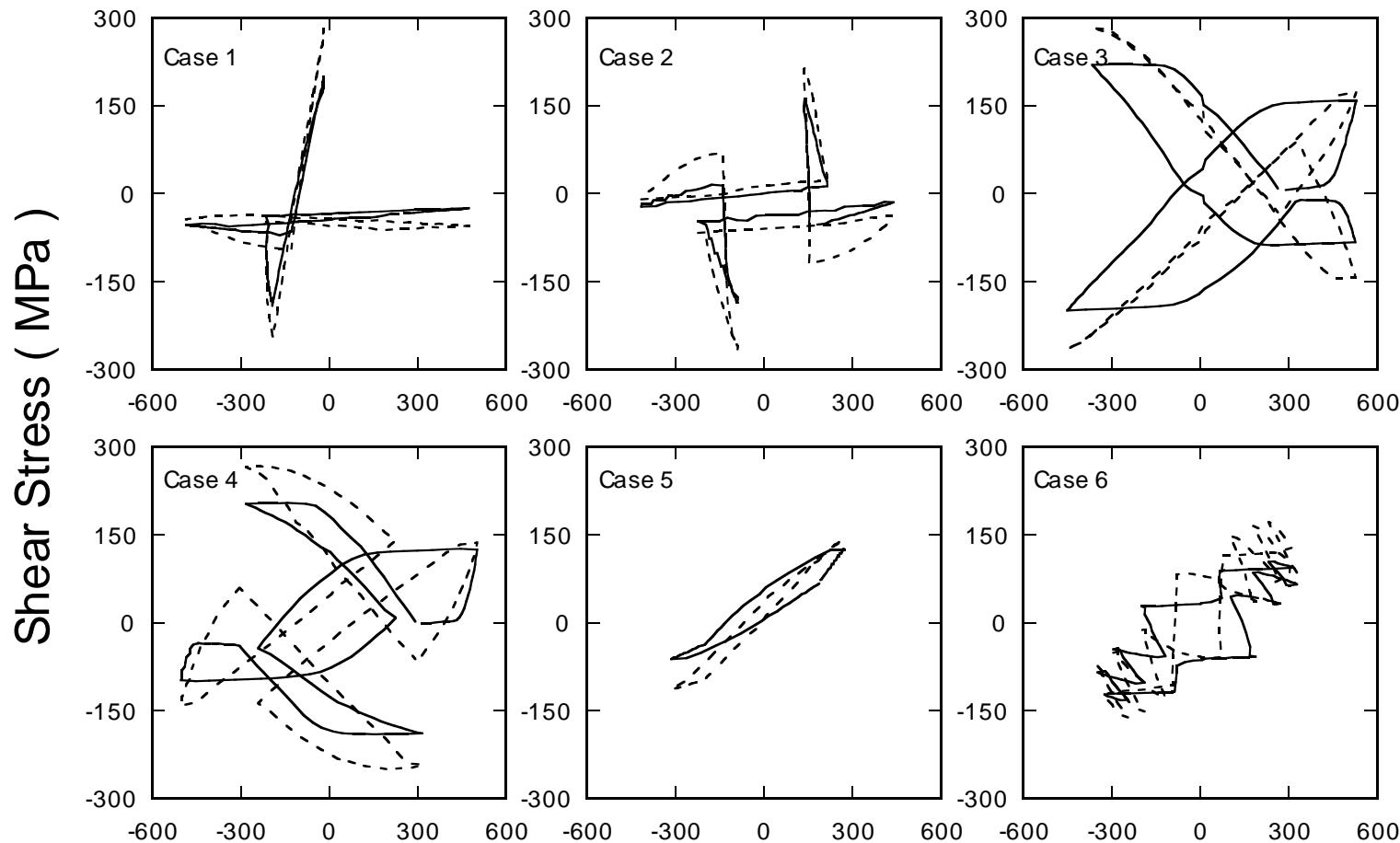
$$\sigma_n \frac{\Delta\varepsilon_1}{2} = \frac{\sigma_f^2}{E} (2N_f)^{2b} + \sigma_f \varepsilon_f (2N_f)^{b+c}$$



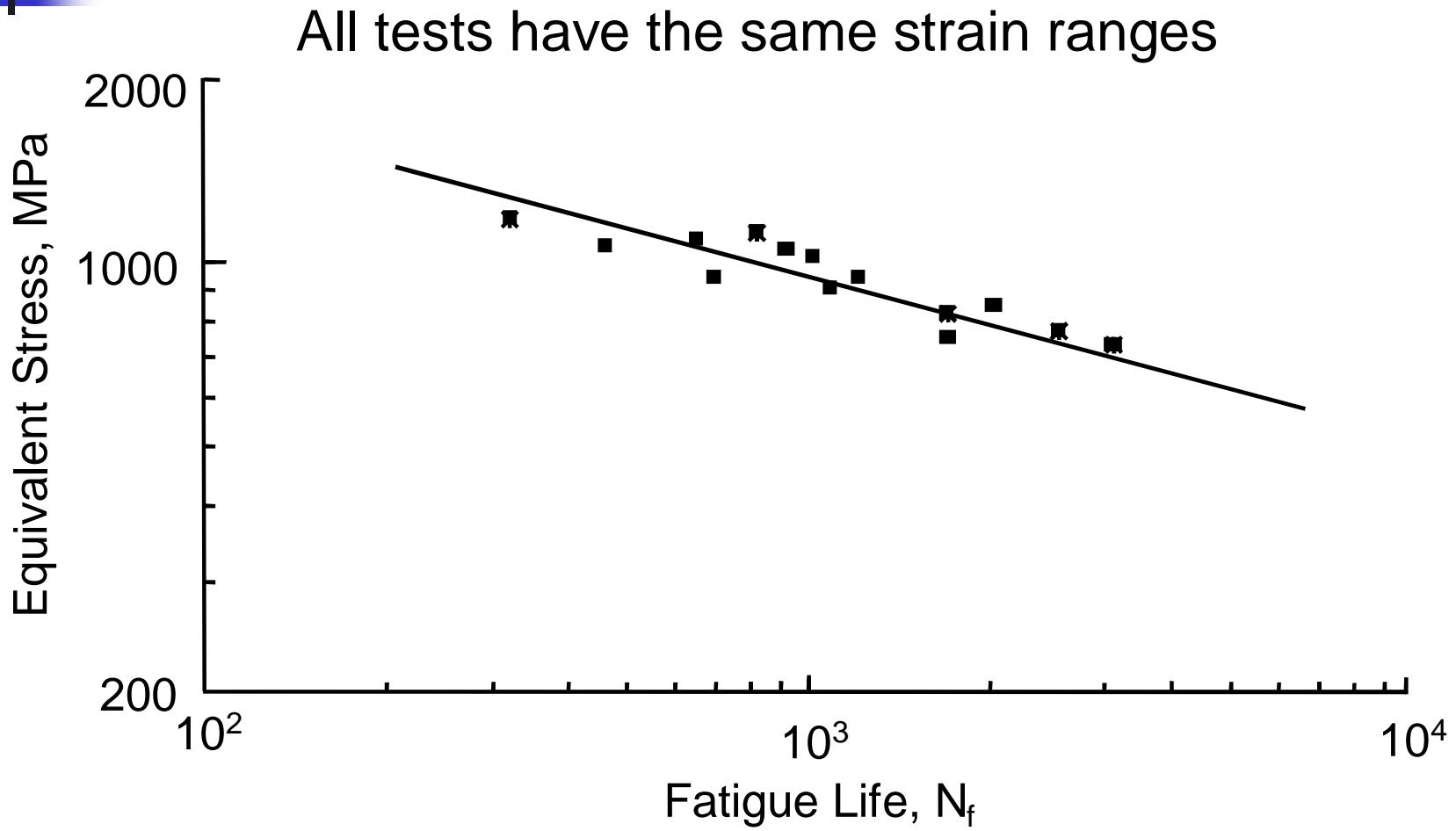
Loading Histories



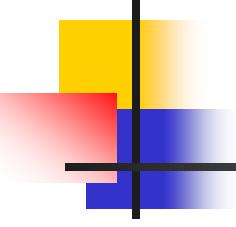
Stress-Strain Response



Maximum Stress

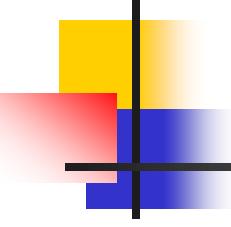


Nonproportional hardening results in lower fatigue lives



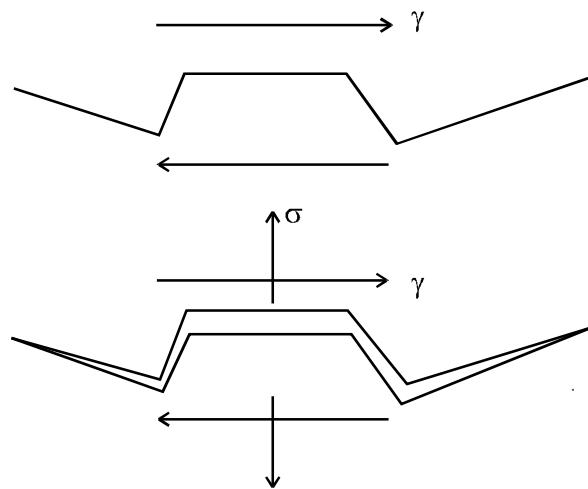
Summary

Cracks nucleate in shear and then grow in either shear or tension depending on the material and state of stress

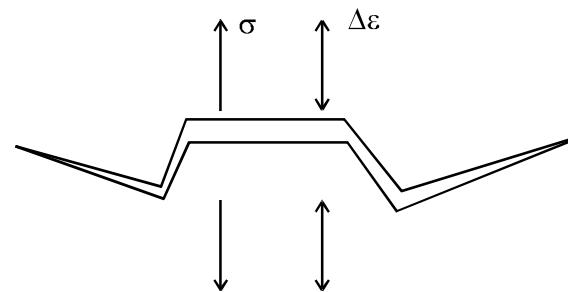


Separate Tensile and Shear Models

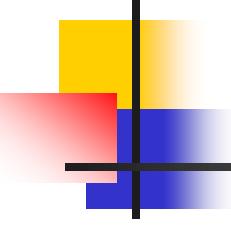
Cyclic shear strains



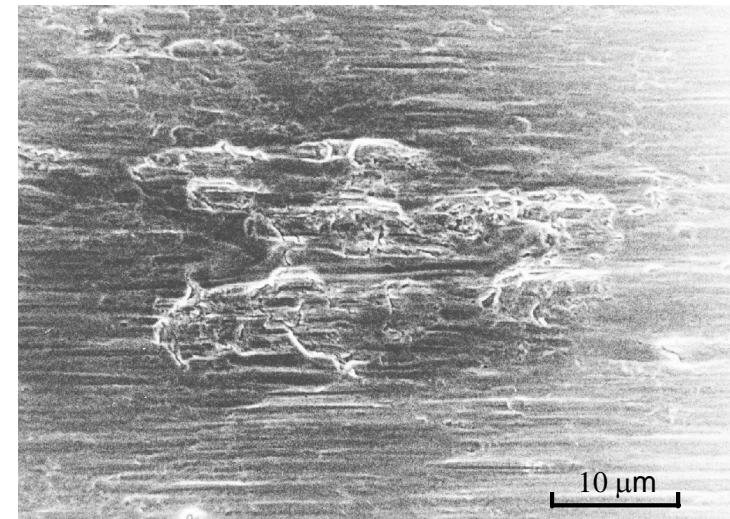
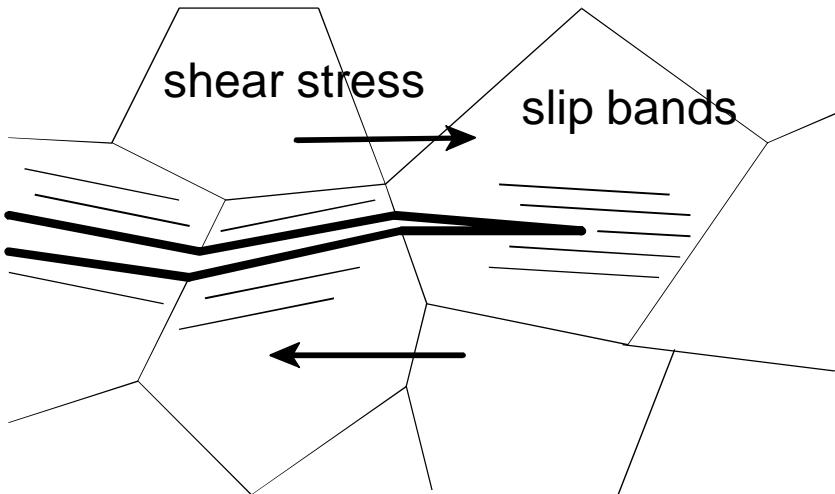
Cyclic tensile strains



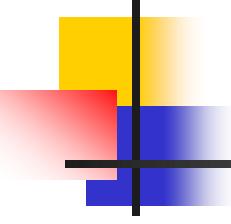
Normal stresses open and close microcracks



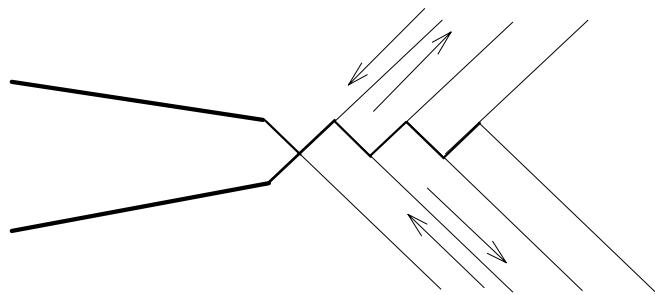
Shear Growth



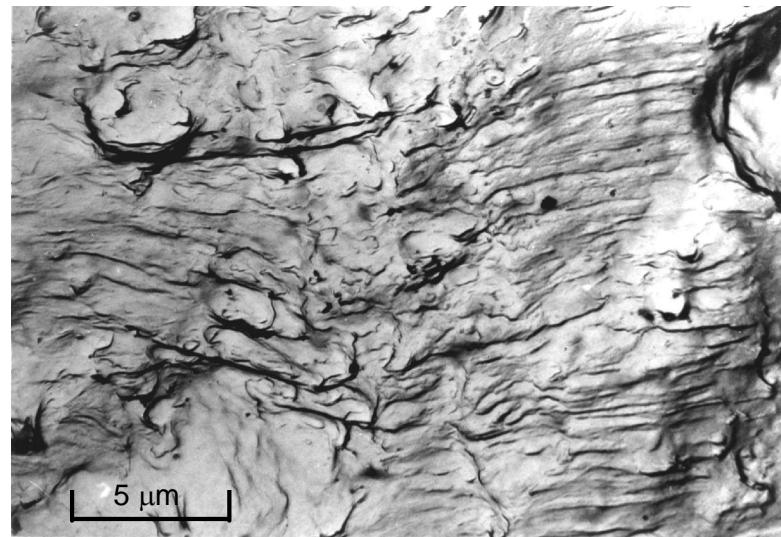
(From Murakami)

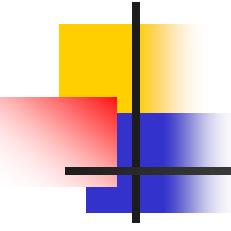


Tensile Growth

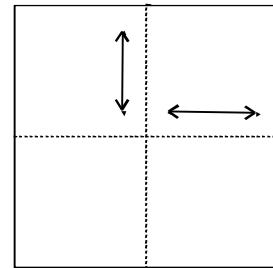
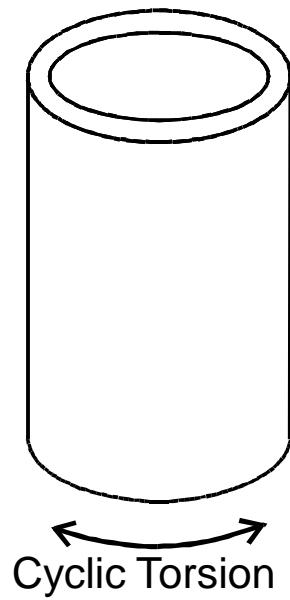


crack growth direction →

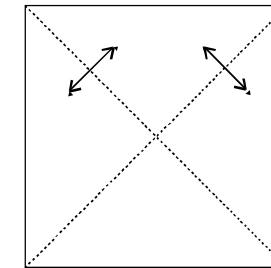




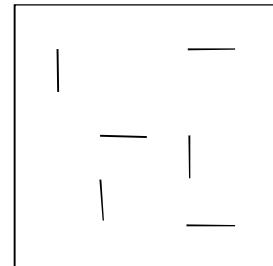
Cyclic Torsion



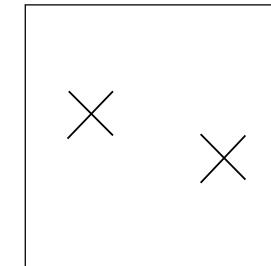
Cyclic Shear Strain



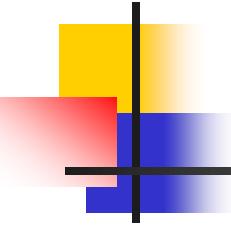
Cyclic Tensile Strain



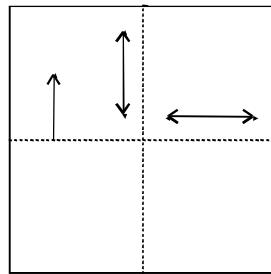
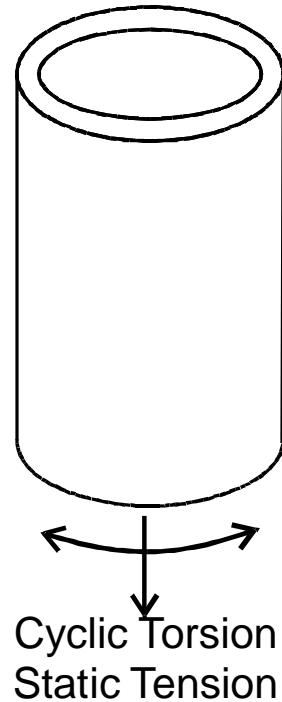
Shear Damage



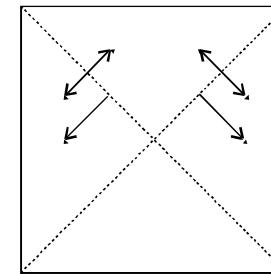
Tensile Damage



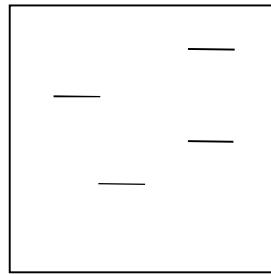
Cyclic Torsion with Static Tension



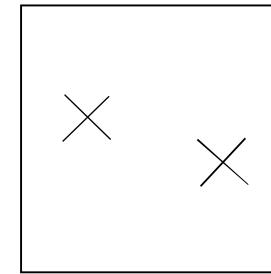
Cyclic Shear Strain



Cyclic Tensile Strain

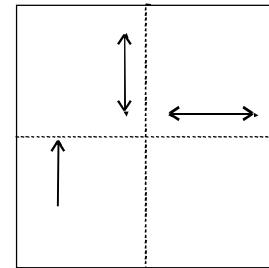
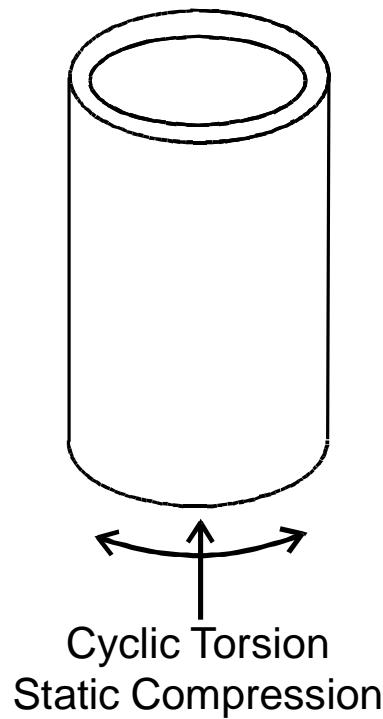


Shear Damage

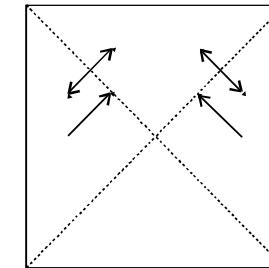


Tensile Damage

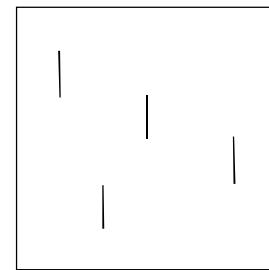
Cyclic Torsion with Compression



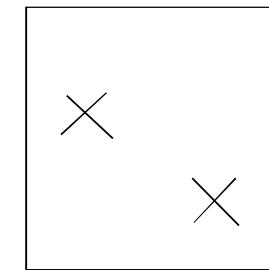
Cyclic Shear Strain



Cyclic Tensile Strain

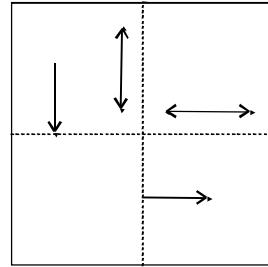
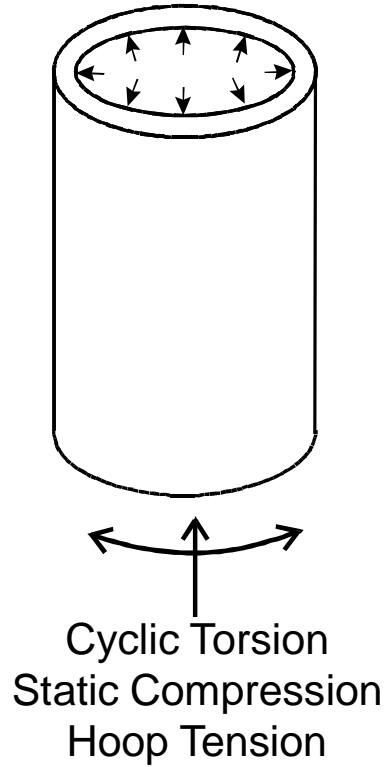


Shear Damage

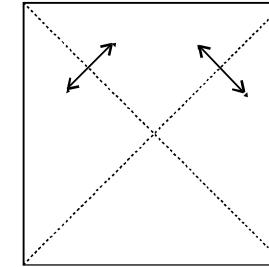


Tensile Damage

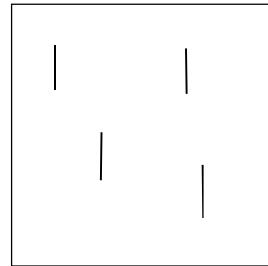
Cyclic Torsion with Tension and Compression



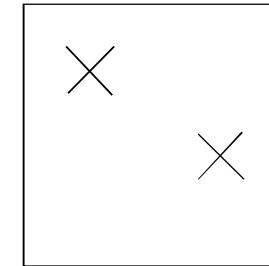
Cyclic Shear Strain



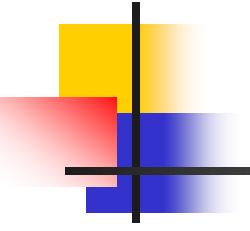
Cyclic Tensile Strain



Shear Damage

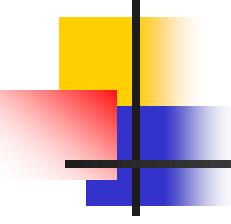


Tensile Damage



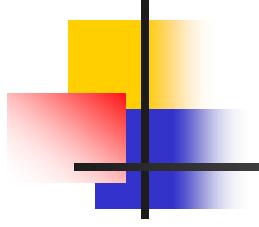
Test Results

Load Case	$\Delta\gamma/2$	σ_{hoop} MPa	σ_{axial} MPa	N_f
Torsion	0.0054	0	0	45,200
with tension	0.0054	0	450	10,300
with compression	0.0054	0	-500	50,000
with tension and compression	0.0054	450	-500	11,200

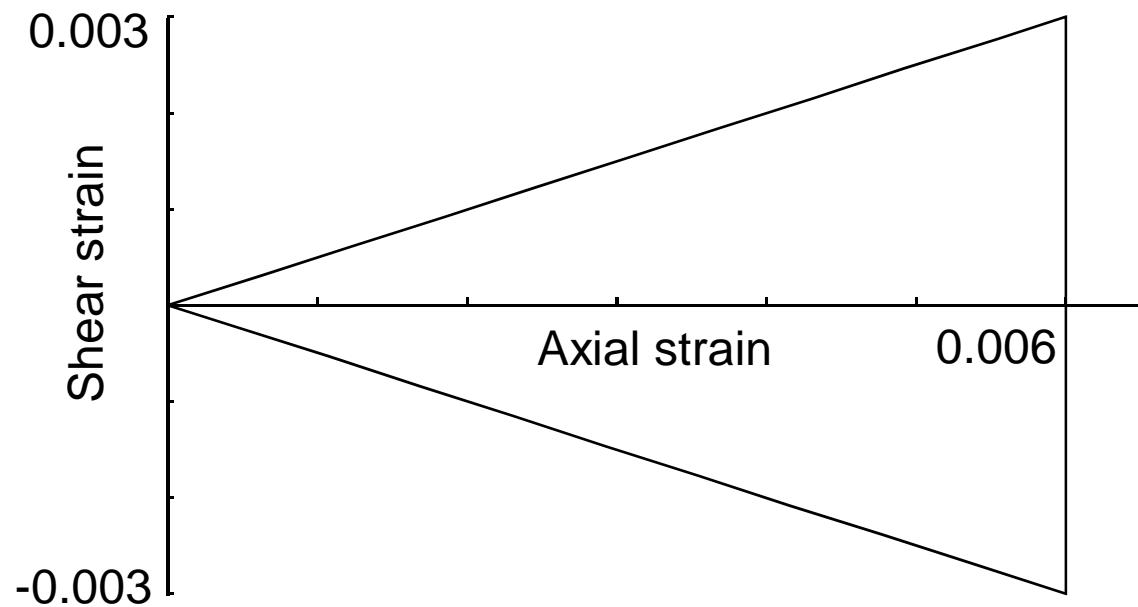


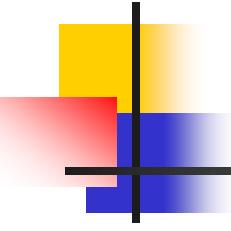
Conclusions

- All critical plane models correctly predict these results
- Hydrostatic stress models can not predict these results



Loading History

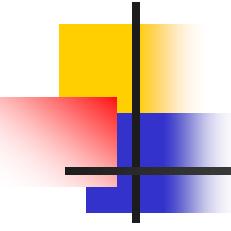




Model Comparison

Summary of calculated fatigue lives

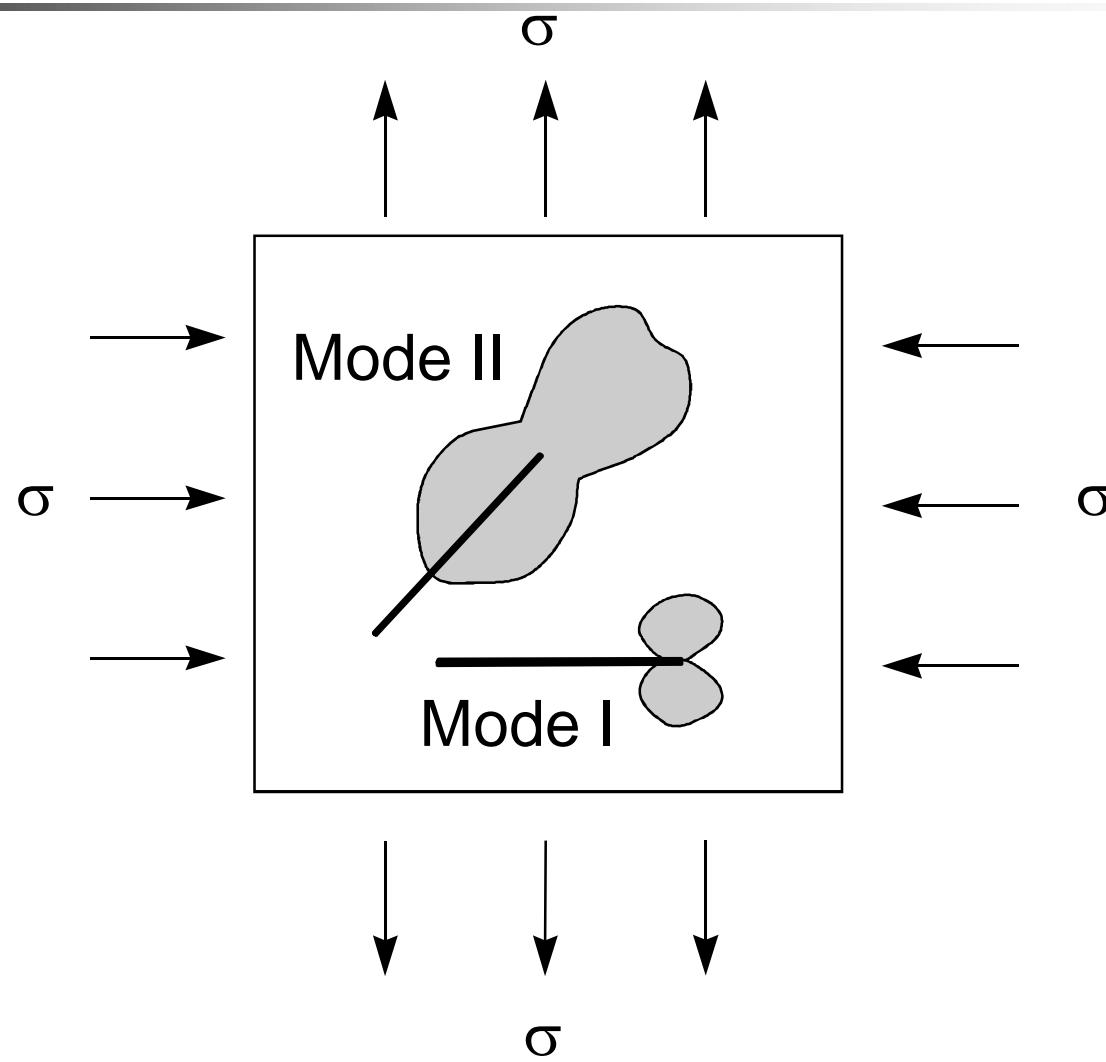
Model	Equation	Life
Epsilon	6.5	14,060
Garud	6.7	5,210
Ellyin	6.17	4,450
Brown-Miller	6.22	3,980
SWT	6.24	9,930
Liu I	6.41	4,280
Liu II	6.42	5,420
Chu	6.37	3,040
Gamma		26,775
Fatemi-Socie	6.23	10,350
Glinka	6.39	33,220



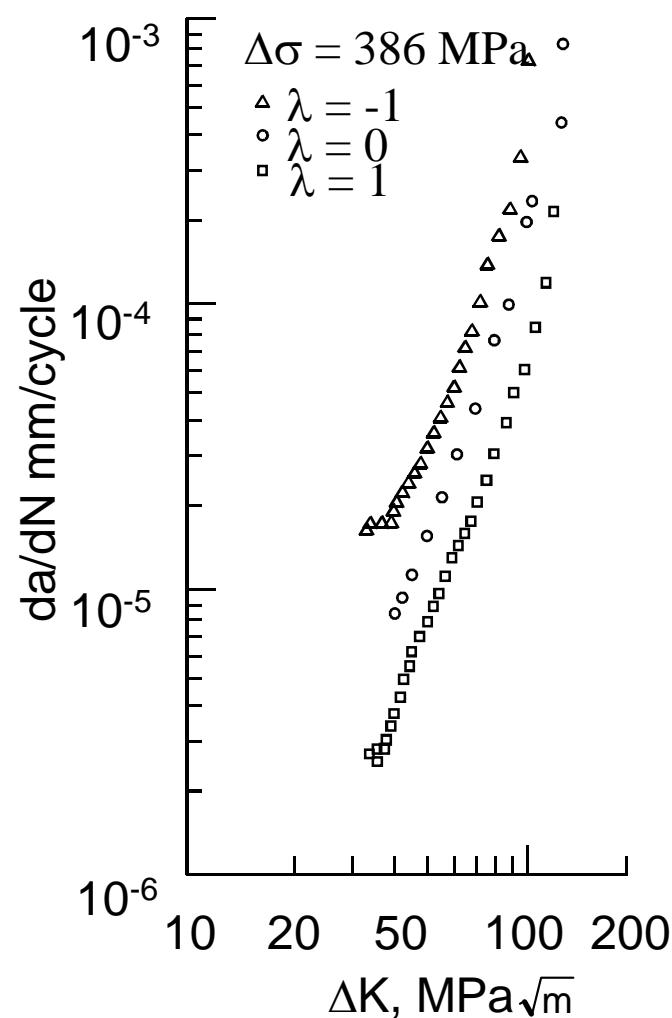
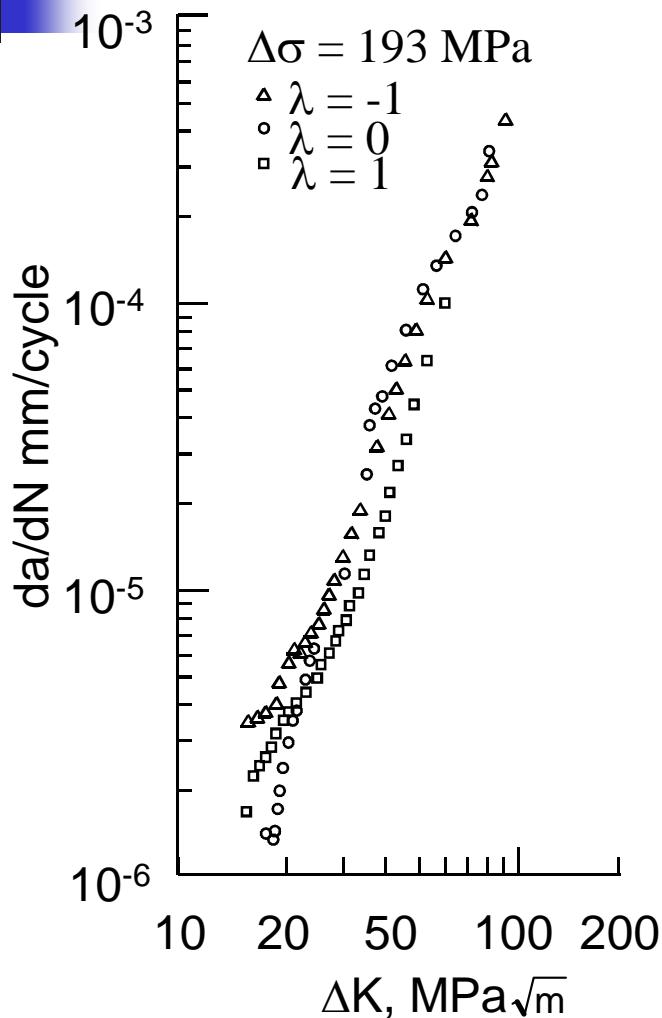
Outline

- Stresses around holes
- Crack Nucleation
- Crack Growth

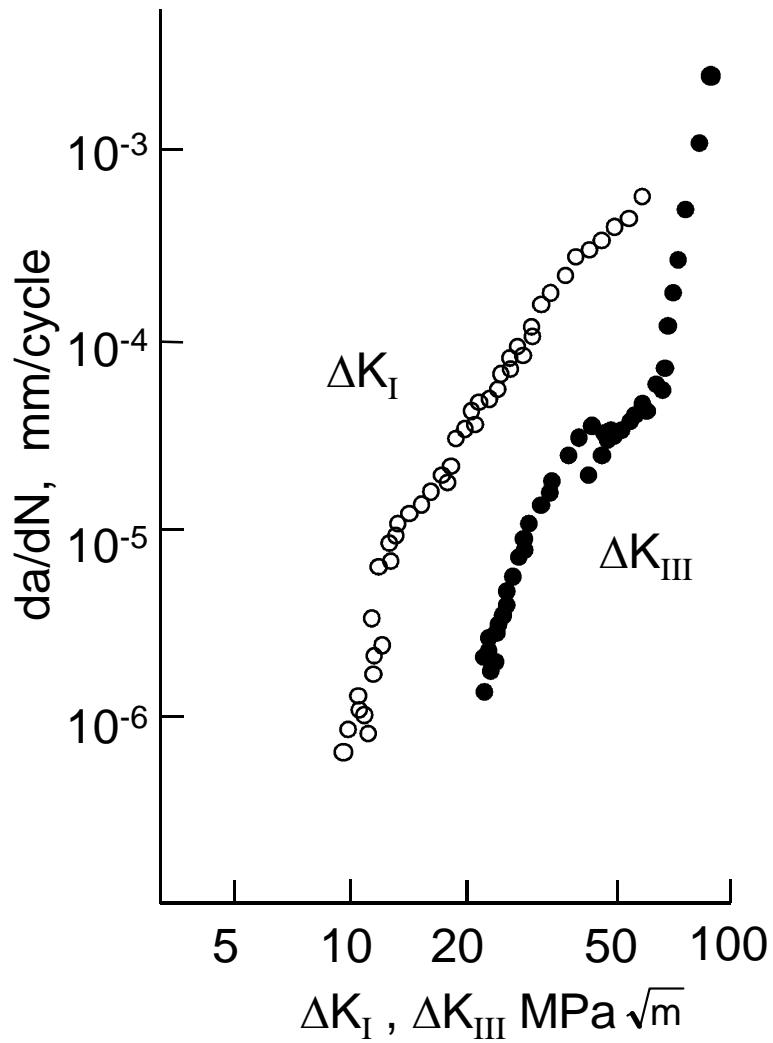
Mode I and Mode II Surface Cracks



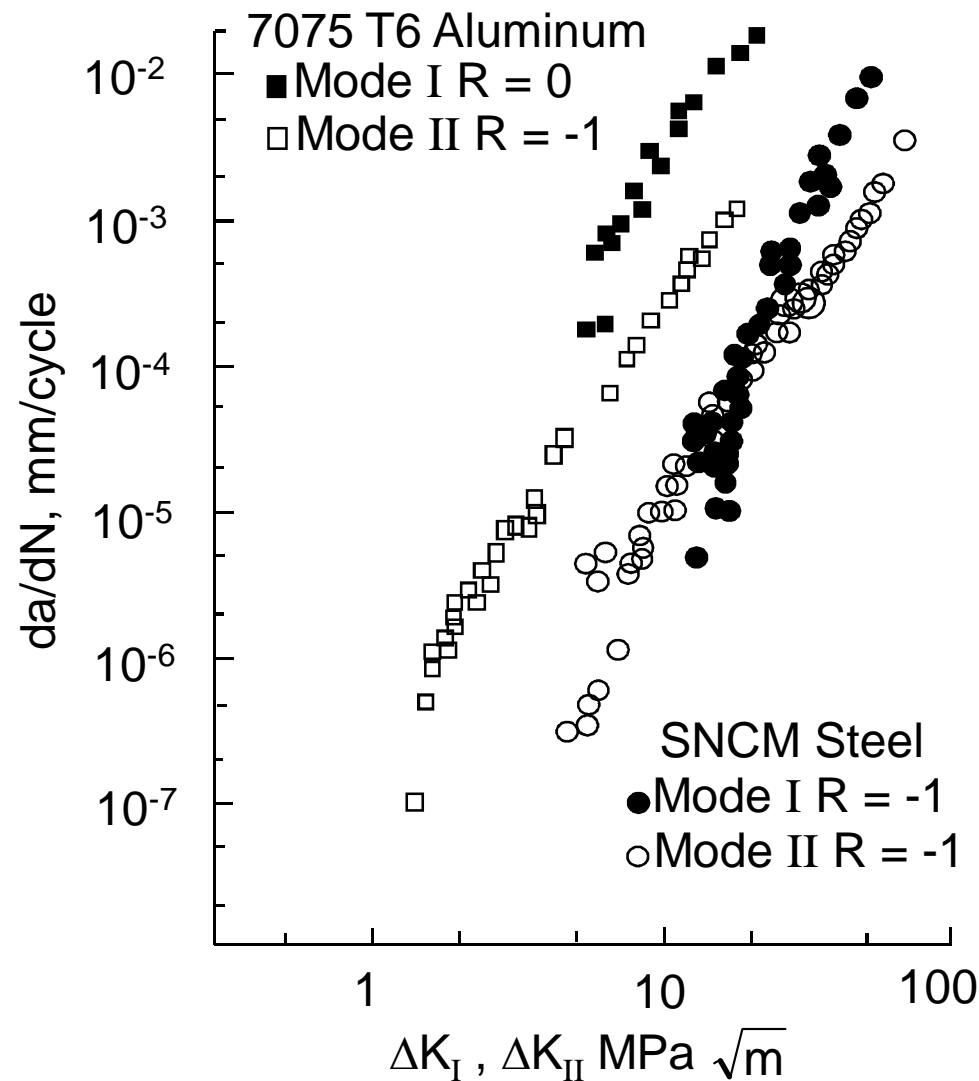
Biaxial Mode I Growth

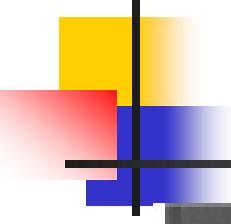


Mode I and Mode III Growth

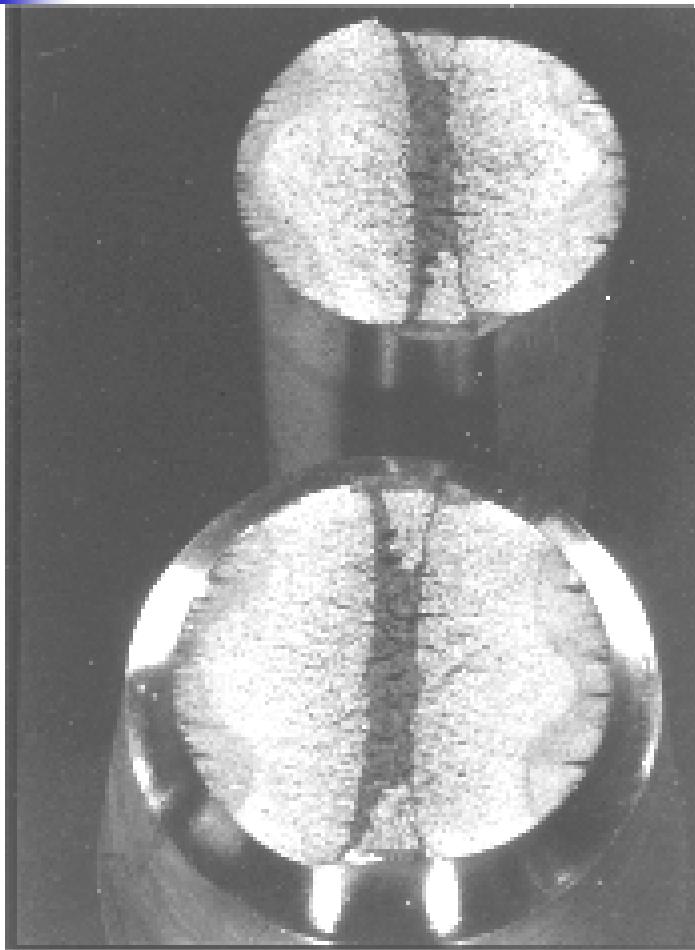


Mode I and Mode II Growth

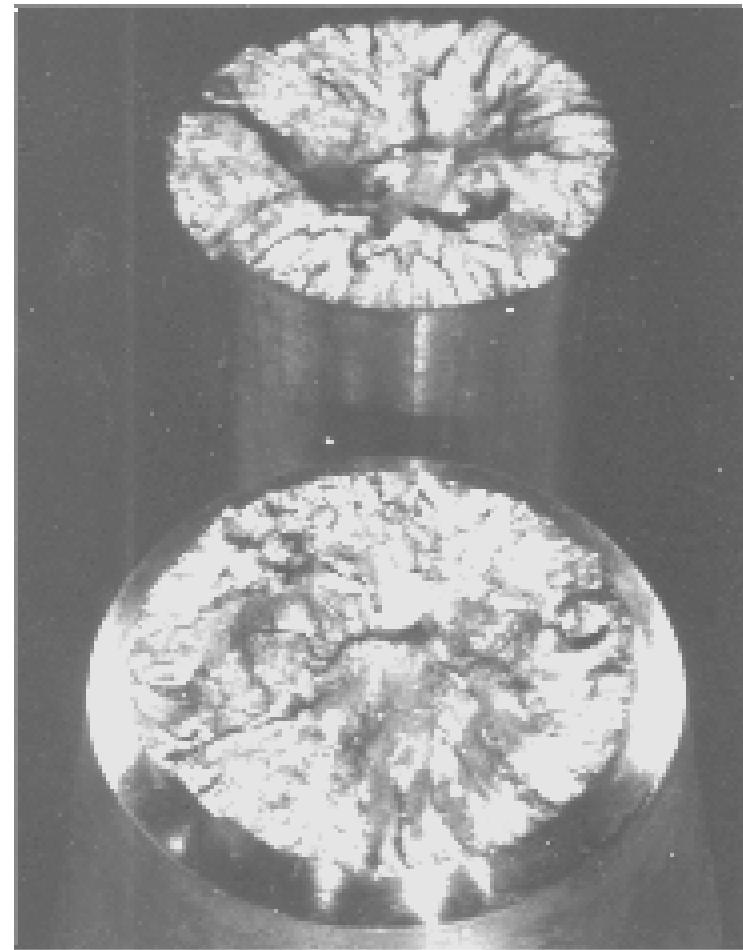




Fracture Surfaces

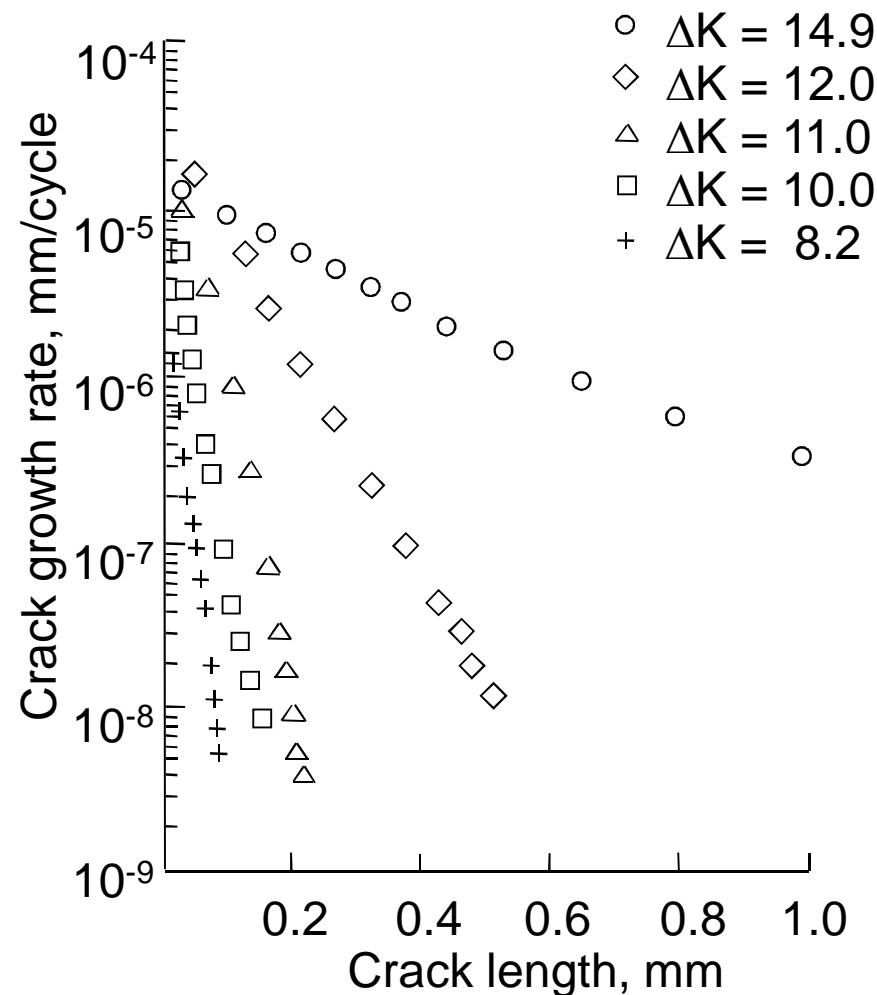


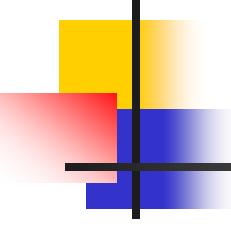
Bending



Torsion

Mode III Growth





Fracture Mechanics Models

$$\frac{da}{dN} = C(\Delta K_{eq})^m$$

$$\Delta K_{eq} = \left[\Delta K_I^4 + 8\Delta K_{II}^4 + 8\Delta K_{III}^4 / (1 - \nu) \right]^{0.25}$$

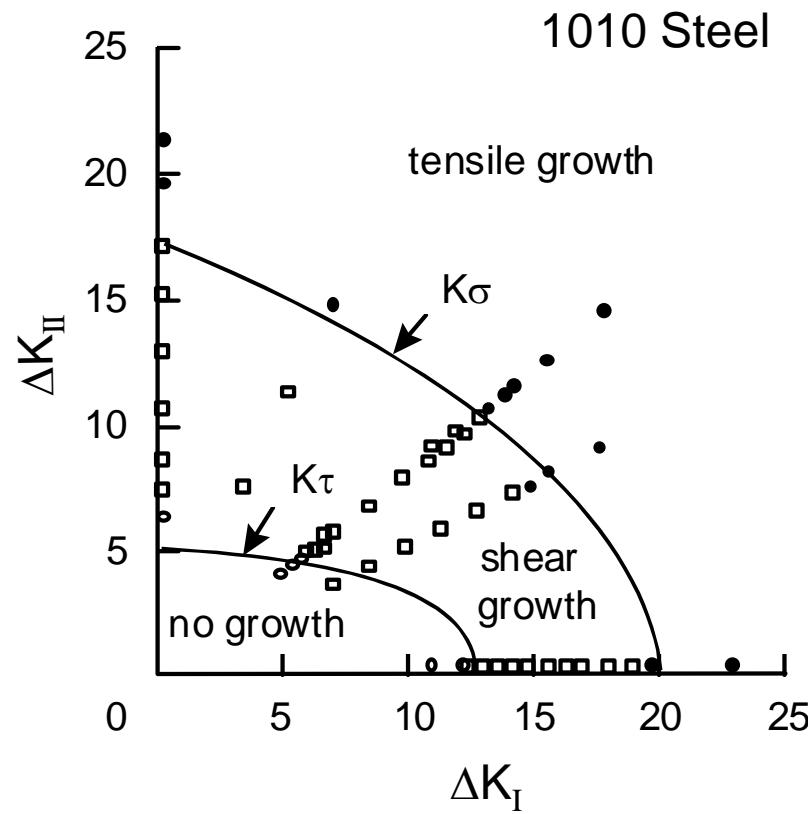
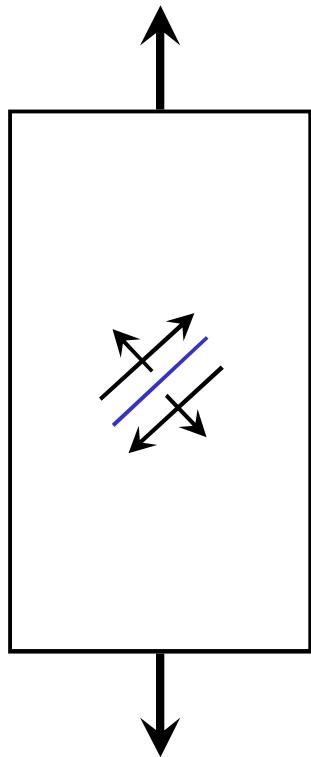
$$\Delta K_{eq} = \left[\Delta K_I^2 + \Delta K_{II}^2 + (1 + \nu) \Delta K_{III}^2 \right]^{0.5}$$

$$\Delta K_{eq} = \left[\Delta K_I^2 + \Delta K_{II}^2 + \Delta K_{III}^2 \right]^{0.5}$$

$$\Delta K_{eq}(\varepsilon) = \left[\left(F_{II} \frac{E}{2(1+\nu)} \Delta \gamma \right)^2 + (F_I E \Delta \varepsilon)^2 \right]^{0.5} \sqrt{\pi a}$$

$$\Delta K_{eq}(\varepsilon) = FG \Delta \gamma \left(1 + k \frac{\sigma_{n,max}}{\sigma_{ys}} \right) \sqrt{\pi a}$$

Growth of Inclined Cracks



Cracks grow in either tension or shear

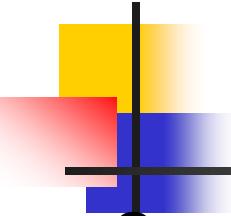
From: Otsuka et.al. *Engineering Fracture Mechanics*, Vol 7, 1975

Tensile growth:

$$K_{\sigma} = \cos \frac{\theta}{2} \left[K_{\parallel} \cos^2 \frac{\theta}{2} - \frac{3}{2} K_{\perp} \sin \theta \right]$$

Shear growth:

$$K_{\tau} = \frac{1}{2} \cos \frac{\theta}{2} \left[K_{\parallel} \sin \frac{\theta}{2} + K_{\perp} (3 \cos \theta - 1) \right]$$



Strain Energy Density

Strain energy density at the crack tip:

$$S = a_{11}K_I^2 + 2a_{12}KK_{II} + a_{22}K_{II}^2 + a_{33}K_{III}^2$$

Necessary and sufficient conditions for crack growth:

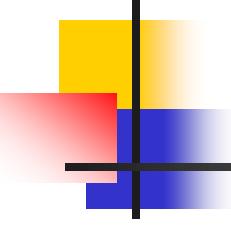
$$\frac{\partial S}{\partial \theta} = 0 \quad \text{at } \theta = \theta_o$$

$$\frac{\partial^2 S}{\partial \theta^2} > 0 \quad \text{at } \theta = \theta_o$$

Cyclic strain energy density:

$$\begin{aligned} \Delta S = 2 & [a_{11}(\theta_o) K_I^{\text{mean}} \Delta K_I + a_{12}(\theta_o) (K_{II}^{\text{mean}} \Delta K_I + K_I^{\text{mean}} \Delta K_{II}) \\ & + a_{22}(\theta_o) K_{II}^{\text{mean}} \Delta K_{II} + a_{33}(\theta_o) K_{III}^{\text{mean}} \Delta K_{III}] \end{aligned}$$

Sih, G.C and Barthelemy, B.M. "Mixed Mode Fatigue Crack Growth Predictions" *Engineering Fracture Mechanics*, Vol. 13, 1980



Summary

Many models but no experimental verification
for out-of-phase spectrum loads

Multiaxial Fatigue

